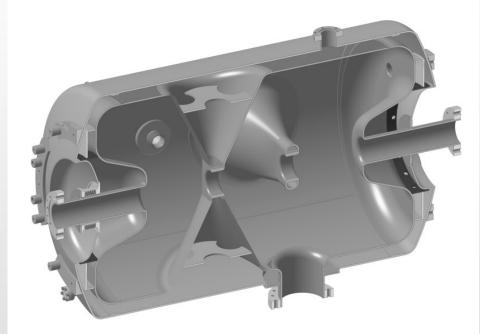




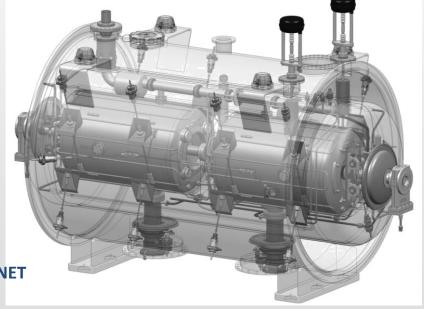
# Design of the 352MHz, beta 0.50, Double-Spoke Cavity for ESS



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Sylvain BRAULT, Sébastien BOUSSON, Patxi DUTHIL, Denis REYNET

Institut de Physique Nucléaire d'Orsay





□ CONTEXT
☐ RF DESIGN OF THE RESONATOR
☐ MECHANICAL DESIGN OF THE RESONATOR
☐ INTEGRATION IN THE CRYOMODULE
☐ STATUS OF THE PROTOTYPES

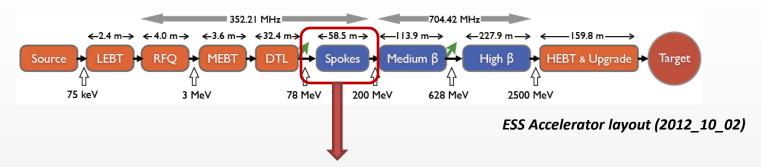


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#### **ESS SPOKE SECTION**

## ☐ ESS Superconducting Spoke section:



- ➤ 28 Double Spoke cavities (3 accelerating gaps)
- ➤ beta=0.50
- > frequency: 352.2 MHz
- > grouped by pair in 14 cryomodules
- > operating temperature: 2K
- Accelerating gradient: Eacc = 8 MV/m
- ➤ Peak field specifications: Epk < 35 MV/m, Bpk < 70 mT

Power c

E. Ramp

THP0





☐ Design

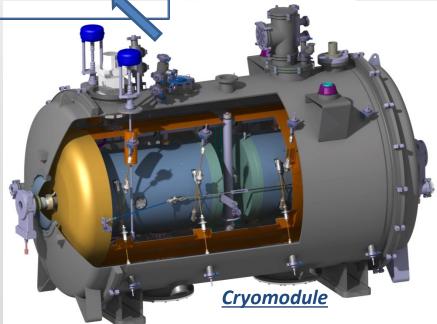
**☐** Fabrication of prototypes

#### **☐** tests of prototypes:

- Vertical tests of cavities
- Power couplers conditioning (Test bench @CEA/Saclay)
- Tests of CTS
- Low power tests of cryomodule (High power tests at UPPSALA)

Gandolfo THP078





**Cold Tuning System** 



□ CONTEXT
☐ RF DESIGN OF THE RESONATOR
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## SPECIFICATIONS FOR THE DOUBLE SPOKE CAVITY

## Parameters established by the beam dynamics simulations:

DOUBLE-SPOKE CAVITY	,
Beam mode	Pulsed (4% duty cycle)
Frequency [MHz]	352.2
Beta_optimal	0.50
Temperature (K)	2
Bpk [mT]	70 (max)
Epk [MV/m]	35 (max)
Gradient Eacc [MV/m]	8
Lacc (=beta optimal x nb of gaps x $\lambda$ /2) [m]	0.639
Bpk/Eacc [mT/(MV/m)]	< 8.75
Epk/Eacc	< 4.38
Beam tube diameter [mm]	50 (min)
RF peak power [kW]	300 (max)



### **OPTIMIZATION OF THE GEOMETRY**

☐ Main goal: fulfil the criteria of the peak surface field to accelerating gradient ratios

$$\frac{E_{pk}}{E_{acc}} < 4.38 \qquad \qquad \frac{B_{pk}}{E_{acc}} < 8.75 \text{ [mT/MV/m]}$$

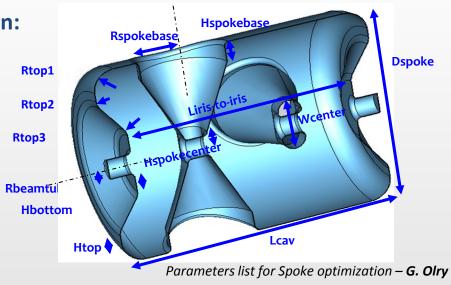
#### ☐ The optimization method of the RF design:

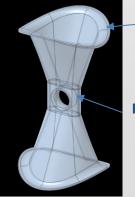
- Parameterization of the geometry
- Sensitivity analysis on the ratios Epk/Eacc & Bpk/Eacc
- CST MicroWave Studio (MWS)
- Results cross-checked with to mesh types: hexahedral and tetrahedral

#### ☐ Geometry of the spoke bars:

Based on our feedback from two Single-Spoke resonators and a Triple-Spoke resonator fabrication (EURISOL)

→ Achievement of an acceptable solution





**Conical shape** 

**Racetrack shape** 



#### **RF RESULTS**

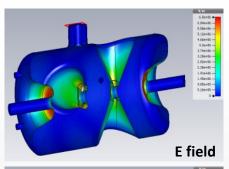
#### Last modifications (included in the prototypes)

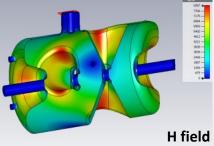
Technical issues for manufacturing

Minor parameters changes

New ESS requirements

Mesh type	Hexahedral (2.2 millions meshcells)	Tetrahedral (600000 tetra.)
Beta optimal	0.50	0.50
Epk/Eacc	4.51	4.33
Bpk/Eacc [mT/MV/m]	6.99	6.89
G [Ohm]	131	130
r/Q [Ohm]	425	426





- Epk/Eacc > 4.38: compromise between the cavity length, end cap shape feasibility and tuning sensitivity.
- Lacc = 3 / 2 x beta optimal x lambda

## □ Coupling calculations: Qext = $1.5 \cdot 10^5$ (with the parameters 50mA and 8MV/m)

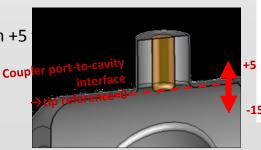
Coupler port location ( $\emptyset$ =100mm):

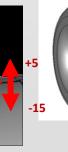
Variation of the coupler port center from 100 to 170mm ( $\leftrightarrow$  distance to the origin)

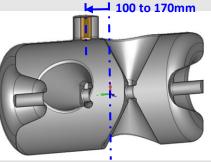
Penetration of the antenna: Variation from +5

Qext =  $1.5 \cdot 10^5$  for:

- $\Rightarrow$  5mm of tip penetration
- ⇒ coupler port location: 120mm







MWS model of the cavity with antenna - G. Olry

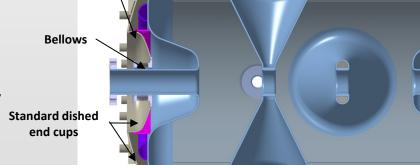


CONTEXT
RF DESIGN OF THE RESONATOR
MECHANICAL DESIGN OF THE RESONATOR
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#### **MECHANICAL STUDIES**

- ☐ Criteria taken into account
  - Cavity preparation: High Pressure Rinsing (HPR)
    - → easy and efficient
  - Life cycle of the cavity: Leak tests & cryomodule tests
    - → No risk of damage (plastic deformation at room T°)
  - Manufacturing constraints: Metal forming & Assembly process
    - → Feasible (at a reasonable cost)
- ☐ Integration of the Helium vessel
- Connections with the beam tubes: Flange / bellows (For the tuning)
- Helium vessel: Titanium grade 2
  - > Ease of assembly with niobium
  - ➤ No problem of thermal stresses
  - May act as a reinforcement of the cavity
- Standard dished end cups



Pick-up ports

Vessel walls thickness = 4mm

⇒ Result of the iterative numerical simulations

ESS Double Spoke Cavity - S. Brault

cavity walls thickness = 4mm



Welded flange

HPR ports (x4)

donut rib (x2)

Ring rib

Helium vessel

Coupler port



#### **MECHANICAL BEHAVIOUR**

#### ■ Mechanical simulations

Different load cases studied according to the life cycle of the cavity

#### Static and modal analysis

(ANSYS Mechanical V14)

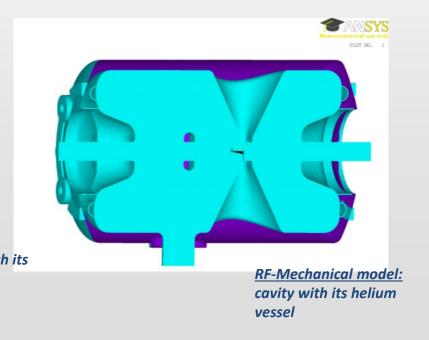
- Leak tests during fabrication
- Pressure test (Cool down at 4K)
- Mechanical vibration modes
- → Check no plastic strains
- → Define maximum pressure during cool down
- → Check sensitivity to microphonics

# Mechanical model: cavity with its helium vessel

#### **RF-Mechanical coupled analysis**

(ANSYS APDL & EMAG V14)

- RF sensitivity by pulling on beam tubes
- RF sensitivity due to the He bath pressure fluctuation
- RF sensitivity due to the Lorentz forces
- → Define sensitivity for the cold tuning system
- → Define a range for the pressure and Lorentz detuning factors

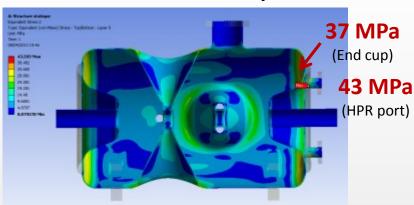




#### STATIC AND MODAL RESULTS

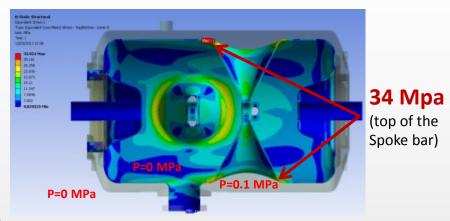
#### **☐** Static results

Leak test on the bare cavity:



- $\Rightarrow$   $\sigma_{\text{max}}$  < 50 MPa (Yield stress of Niobium at room T°)
- ⇒ The donut ribs are necessary

#### Pressure test with $\Delta P = +0.1$ MPa:

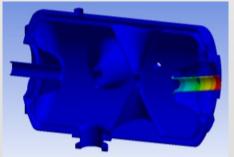


 $\Longrightarrow$  Max pressure (Cool down) estimated to be 1.47 bar at  $\sigma_{\text{max}}$  = 50 Mpa

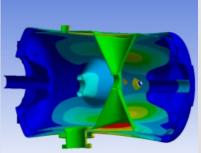
#### **☐** Mechanical modes

N°	Frequency	Mode
1 & 2	212Hz	Beam tube on CTS side
3 & 4	265Hz & 275Hz	Spoke bar/Helium vessel
5 & 6	285Hz	Coupled mode Cavity/Helium vessel
7	313Hz	Helium vessel
8 to 11	315Hz to 365Hz	Coupled mode Cavity/Helium vessel
12	396Hz	beam tubes

⇒ First critical mode (mode 3) >> 50 Hz



Mode 1: 212 Hz



Mode 3: 265 Hz



#### **RESULTS ON THE RF SENSITIVITY**

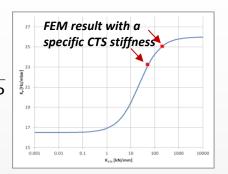
#### ☐ Sensitivity to Helium bath pressure fluctuation

K <sub>P</sub> without CTS (free ends)	+16.5 Hz/mbar
K <sub>P</sub> with greatly stiff CTS*	+26.0 Hz/mbar

\*The beam tube is connected rigidly to the helium vessel at the level of the 4 CTS supports (along the beam axis)

K<sub>P</sub> as a function of the CTS stiffness:

$$K_{p} = K_{p}^{\infty} - \frac{\partial f}{\partial z} \frac{\vec{F}^{\infty} \cdot \vec{u}_{z}}{(K_{cav} + K_{CST})\Delta P}$$



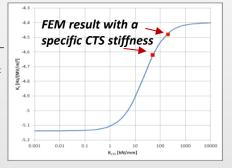
#### ☐ Sensitivity to Lorentz forces detuning

		For 8MV/m
K <sub>L</sub> without CTS (free ends)	-5.13 Hz/(MV/m) <sup>2</sup>	Δf = -328 Hz **
K <sub>L</sub> with stiff CTS	-4.4 Hz/(MV/m) <sup>2</sup>	∆f = -282 Hz

\*\*bandwidth = 1530 Hz

 $K_L$  as a function of the CTS stiffness:

$$K_{L} = K_{L}^{\infty} + \frac{\partial f}{\partial z} \frac{\vec{F}^{\infty}.\vec{u}_{z}}{(K_{cav} + K_{SAF})E_{acc}^{2}}$$



## ☐ RF sensitivity for cavity tuning

Stiffness of the cavity	20 kN/mm
Tuning sensitivity ∆f/∆z	135 kHz/mm

 $\Rightarrow$  At 2K: the tuning range is +173 kHz (1.28mm of max displacement not to exceed 400 MPa)



View of the Cold Tuning System - N. Gandolfo

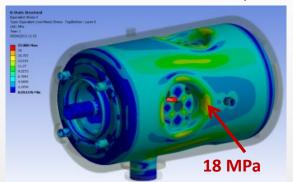


#### **DETAILED MECHANICAL STUDIES**

#### ☐ Last modifications (included in the prototypes)

Adding of some new stiffeners on the Spoke bars:

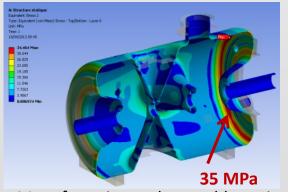
Pressure test with  $\Delta P = 0.1$  Mpa:



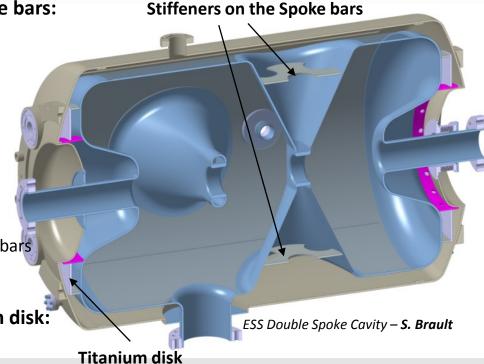
⇒ Maximum pressure (Cool down) estimated to 2.77 bars

Replacement of the donut rib by a titanium disk:

Leak test on the bare cavity:



⇒ Manufacturing and assembly easier



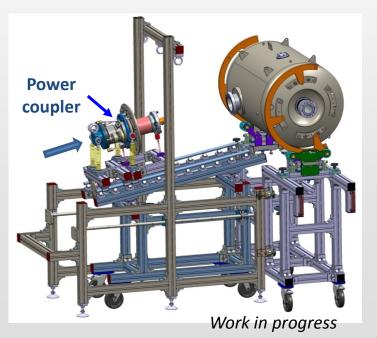


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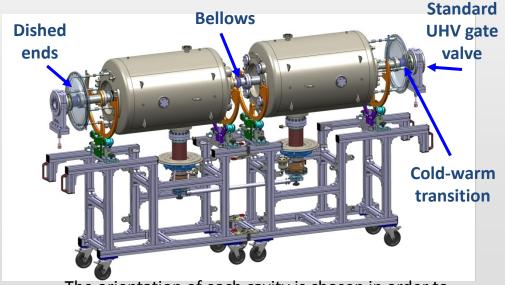


### **CAVITY ASSEMBLY INTO THE CLEAN ROOM**

- ☐ High Pressure Rinsing HPR (100bars) in clean room ISO 4
- ☐ Assembly of the cavities with:
  - power coupler
  - > cold-warm transitions, dished ends and bellows
  - > warm Ultra High Vacuum gate valves





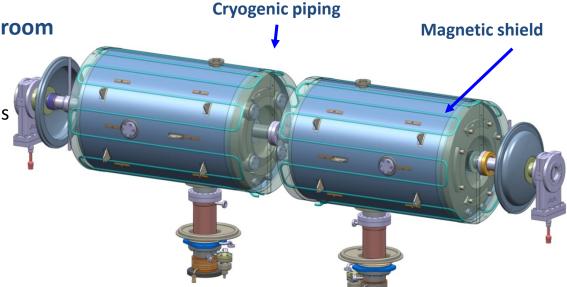




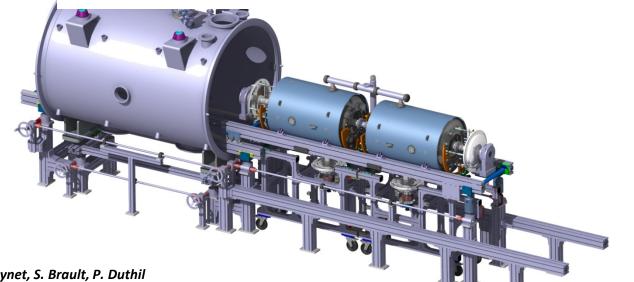
### **ASSEMBLY OUTSIDE THE CLEAN ROOM**

#### ■ Assembly outside the clean room

- ➤ Magnetic shield
- > Cryogenic distribution
- > Thermal shield and supporting rods
- Cold tuning system ...



**☐** Tooling for cryostating



D. Reynet MOP089

ESS Spoke Cryomodule – D. Reynet, S. Brault, P. Duthil

Details in: "Design of the ESS Spoke cryomodule", SRF 2013, these proceedings.



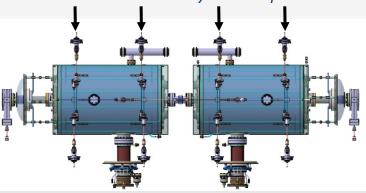
#### **SUPPORTING SYSTEM**

#### ☐ Principle of supporting system

#### **Several considerations:**

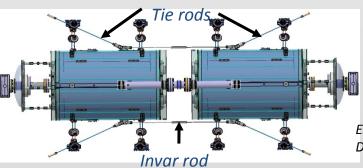
- 2 cavities: length = 2.86m, weight <500 Kg (with the
- Static heat load
- Assembly and alignment methods
- > Antagonist tie rods in some vertical planes
  - → Vertical and lateral positions

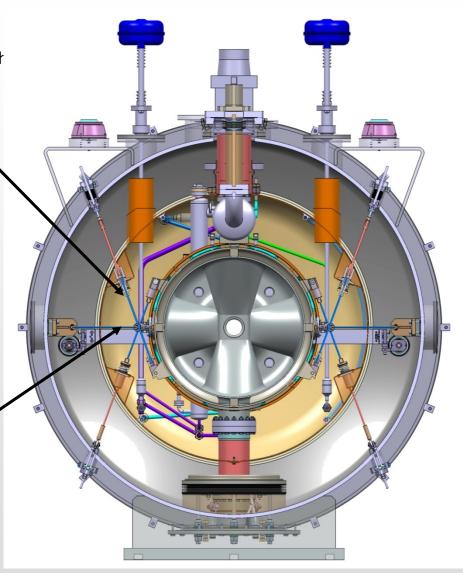
4 identical tie rods by vertical plane



> Tie rods and invar rods in a horizontal plane

→ Position along beam axis





ESS Spoke Cryomodule – **D. Reynet, S. Brault, P. Duthil**Details in: "Design of the ESS Spoke cryomodule", SRF 2013, these proceedings.



☐ STATUS OF THE PROTOTYPES
☐ INTEGRATION IN THE CRYOMODULE
☐ MECHANICAL DESIGN OF THE RESONATOR
☐ RF DESIGN OF THE RESONATOR
□ CONTEXT



#### **FABRICATION OF PROTOTYPES**

#### ☐ Cavity: 3 prototypes

- 1 by SDMS (France)
- 2 by ZANON (Italy)
- > Start of contract: March 2013
- ➤ Ongoing discussions about the manufacturing of:
  - the Spoke bars in several pieces
  - the end cups of the cavity
- ➤ Delivery: April 2014

#### **☐** Power coupler: 4 prototypes

- 2 by SCT (France)
- 2 by PMB (France)
- > Start of manufacturing: September 2013
- ➤ Delivery: November 2013

#### ☐ Cold Tuning System: 2 prototypes

ESIM (France): mechanical components
NOLIAC (Denmark) &
PHYSIK INSTRUMENTE (Germany): Piezo actuators

➤ Delivery: done

N. Gandolfo THP078



Prototype mounted on the triple Spoke cavity (Eucard) at IPNO





## THANK YOU FOR YOUR ATTENTION