

THERMAL AND MAGNETIC STRESS IN THE HORN: STATIC CASE

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OUTLINE

- Model: electrical/resistive heating, Magnetic field/magnetic pressure, Temperature field/ thermal stress
- Material properties: electrical and thermal conductivity function of temperature
- Magnetic stress
- Thermal stress
- Total stress
- Fatigue limit
- Total stress with increased cooling or thickness.
- Conclusion

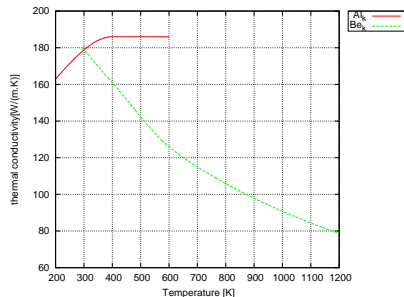
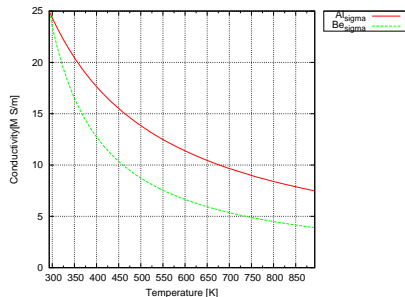
COUPLED PHYSICS MODELS

Model	Equation	Input	BC	Output
AC/DC	$j\omega\mu\mathbf{H} + \frac{1}{\sigma+j\omega\epsilon}\nabla\times[\nabla\times\mathbf{H}] = 0$ $\sigma = \sigma(T)$	$H_{0\phi} = \frac{I_{rms}}{2\pi r}$	$\mathbf{n} \times \mathbf{E} = 0 \Leftrightarrow H_n = 0$	\mathbf{J}, \mathbf{B} Q_{avemqh}
Thermal	$\nabla \cdot [k \nabla T] + q = 0$ $k = k(T)$	$q = Q_{beam} + Q_{avemqh}$	$q'' = \bar{h}[T - T_{\infty}]$	T
Mechanical linear elast	$\frac{\partial\sigma_r}{\partial r} + \frac{\partial\tau_{rz}}{\partial z} + \frac{\sigma_r - \sigma_{\theta}}{r} + F_r = 0$ $\frac{\partial\tau_{rz}}{\partial r} + \frac{\partial\sigma_z}{\partial z} + \frac{\tau_{rz}}{r} + F_z = 0$ $\vec{\sigma} = \mathbf{E}\vec{\epsilon}$	$dF_r = -Re(B_{\phi}) \times Re(J_z)$ $dF_z = Re(J_r) \times Re(B_{\phi})$ $\Leftrightarrow p(r) = \frac{\mu I_0^2}{8\pi^2 r^2}$	$u_r(z=0) = 0$ $u_z(r=0) = 0$	\mathbf{u} \mathbf{s}
Mechanical & thermal	idem $\vec{\epsilon} = \epsilon_{el} + \epsilon_{th}$ $\epsilon_{th} = \mathbf{I}\alpha(T - T_{ref})$	idem α, T	idem $T_{ini} = T_{ref}$	\mathbf{u}_{tot} \mathbf{s}_{tot}

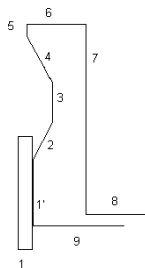
- $I_0 = 350kA$, $I_{rms} = 8750A$. To model total stress, assume a magnetic pressure corresponding to peak current I_0 .
- $Q_{beam} = 55kA$ deposited in the Beryllium target of length $L = 0.78m$ and radius $R = 15$ mm.(obtain with Fluka).
- Cooling: $\{h_{target}, h_{horn}\} = \{10 - 20, 1 - 2\} kW/(m^2K)$
- non linear because both electrical and thermal conductivity are temperature dependant.
- axisymmetric model: all variables are function of r and z.

MATERIAL PROPERTIES

- Model 1: constant electrical and thermal conductivity for Al and Be
- Model 2: Temperature dependant electrical and thermal conductivity for Al and Be



RESISTIVE LOSSES



Q[kW]	tot	1+1''	2	3	4	5 + 6	7	8	9
$\sigma = \sigma_0$	27	14	2.5	1.0	2.6	4.1	1.3	0.23	1.4
$\sigma = \sigma(T)$	37	20.8	2.7	1.0	2.9	6.5	1.3	0.23	1.5

- Total electrical losses are 37% higher than the one calculated with constant electrical conductivity
- Most electrical losses come from the inner conductor, conical sections and top end of the horn.
- $q_{elec} = \frac{\rho}{2} J^2$, the resistivity increased with temperature, \Rightarrow essential to maintain the inner conductor at low temperature.

MAGNETIC FLUX DISTRIBUTION

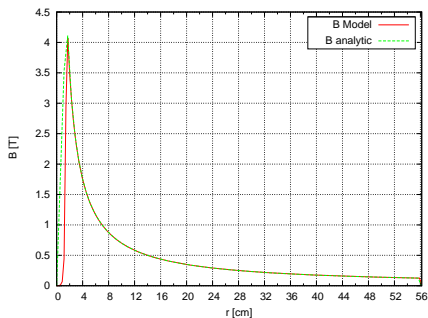
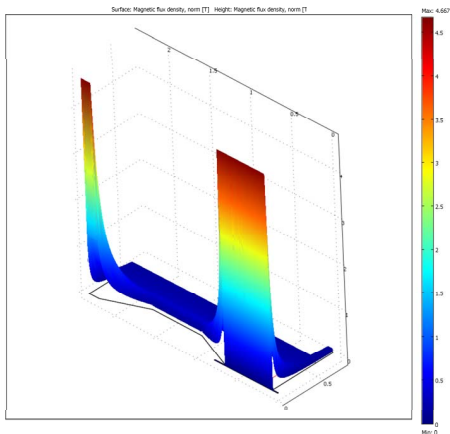
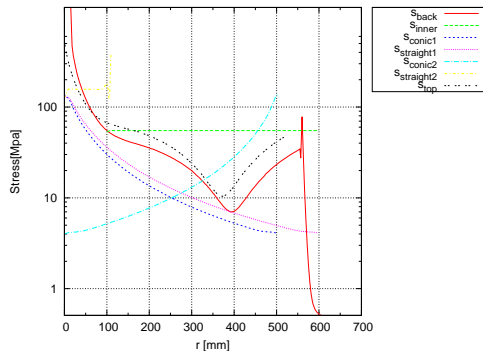


FIGURE: Magnetic flux distribution

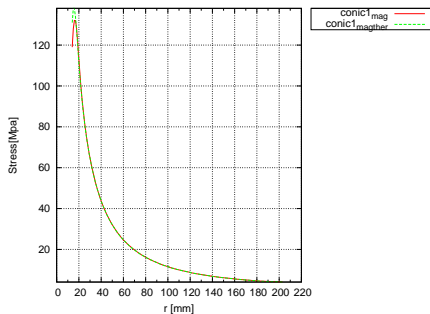
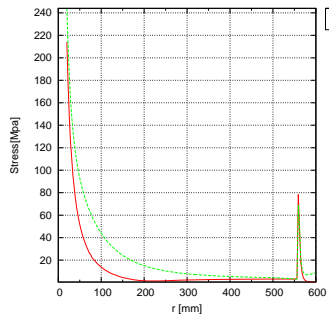
FIGURE: Radial magnetic flux distribution, analytic and model

STRESS FROM MAGNETIC PRESSURE IN THE HORN



- Stress in the conductor around 100 Mpa
- Stress increase with smaller radius

MAGNETIC STRESS IN THE BACK AND CONICAL CONDUCTOR



COMPARISON THERMAL/MAGNETIC STRESS

- magnetic stress is dominant, peak stress corresponding to $I_0 = 350$ kA, frequency: 12.5 Hz.
- thermal stress important for domain with high temperature
- can increase thickness to lower the total stress

TEMPERATURE FIELD, $\sigma, k = Cste$

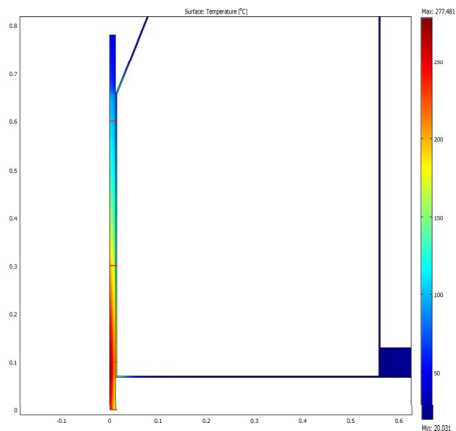


FIGURE: Target and horn, T_{max} above 270°C , $h_{target} = 10\text{kW}/\text{m}^2\text{K}$, $h_{horn} = 1\text{kW}/\text{m}^2\text{K}$

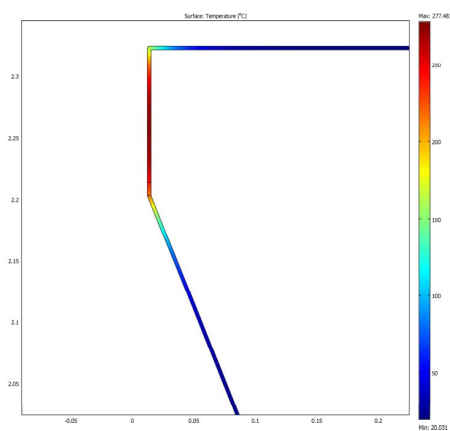


FIGURE: Top end of the horn, T_{max} above 270°C , $h_{target} = 10\text{kW}/\text{m}^2\text{K}$, $h_{horn} = 1\text{kW}/\text{m}^2\text{K}$

TEMPERATURE FIELD, $\sigma(T)$, $k(T)$

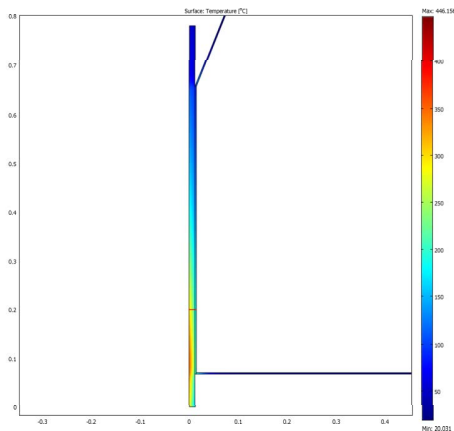


FIGURE: Target and horn, T_{max} around $350\text{ }^{\circ}\text{C}$, $h_{target} = 10\text{ kW/m}^2\text{K}$, $h_{horn} = 1\text{ kW/m}^2\text{K}$

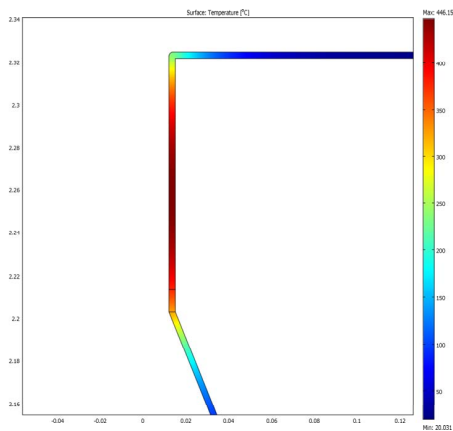


FIGURE: Top end of the horn, T_{max} above $440\text{ }^{\circ}\text{C}$, $h_{target} = 10\text{ kW/m}^2\text{K}$, $h_{horn} = 1\text{ kW/m}^2\text{K}$

TEMPERATURE FIELD, $\sigma(T)$, $k(T)$

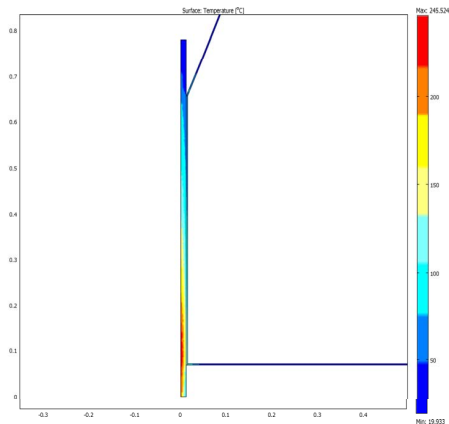


FIGURE: Target and horn, T_{max} around 245°C , $h_{target} = 20\text{kW}/\text{m}^2\text{K}$, $h_{horn} = 2\text{kW}/\text{m}^2\text{K}$

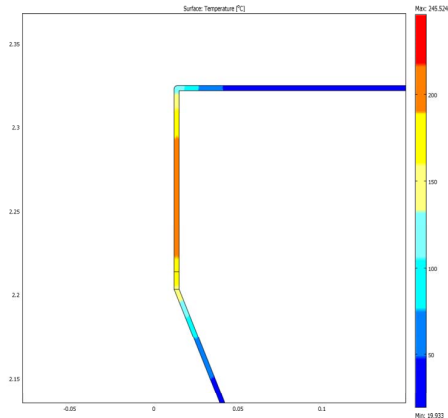


FIGURE: Top end of the horn, T_{max} above 200°C , $h_{target} = 20\text{kW}/\text{m}^2\text{K}$, $h_{horn} = 2\text{kW}/\text{m}^2\text{K}$

ELECTRICAL CONDUCTIVITY

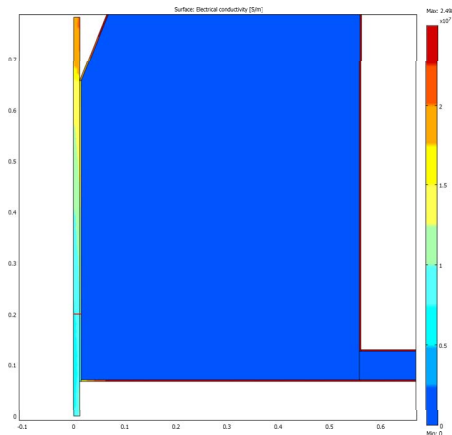


FIGURE: Target and horn,
 $\sigma_{max} = 2.5E7[S/m]$

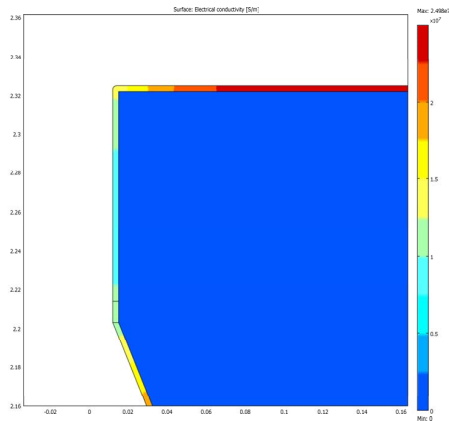
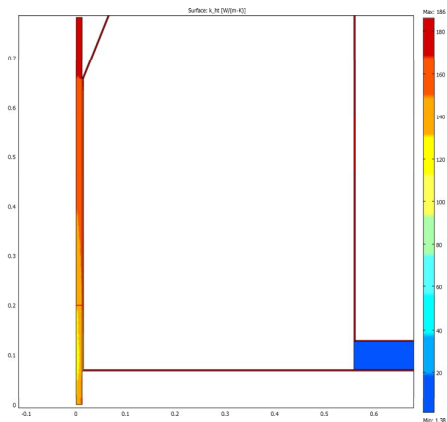


FIGURE: Top end of the horn,
 $\sigma_{max} = 2.5E7[S/m]$

THERMAL CONDUCTIVITY



- thermal conductivity of Al do not vary significantly for Al
- thermal conductivity of Be strongly with temperature \Rightarrow maintain low temperature

TARGET STRESS, σ , $k = cste$

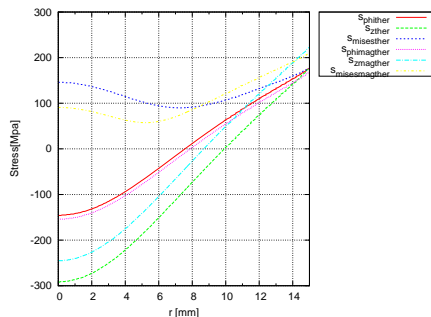
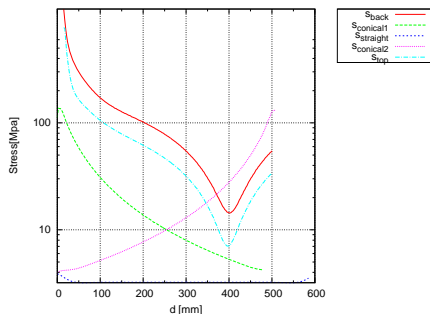


FIGURE: Stress components in the target $z = 2\text{cm}$, thermal and mag+thermal stress

- cooling target $h = 10 \text{ kW}/(\text{m}^2\text{K})$
- $S_\phi = S_r = p(R) = -8.66 \text{ MPa}$.
Model correct checked with analytic expression
- $|S_{\phi mag}| \ll |S_{\phi ther}|$
- $S_{z mag+ther} = S_{z ther} + 46 \text{ Mpa}$
- cylinder in compression in the z direction for $r \in [0, 1] \text{ cm}$
- cylinder in traction in z direction for $r \in [1, 1.5] \text{ cm}$
- Von mises stress level:
 $\sim 100 - 200 \text{ Mpa}$
- Fatigue strength of Beryllium
 $\sim 100 \text{ Mpa}$

STRESS IN THE CONDUCTORS, $\sigma, k = cste$



- Stress gets very high for low radius
- high stress level in perpendicular junction, stress concentrations; singularities.
- Important stress level $\gg 100$ Mpa in back; top and conical sections; especially at low radius.

FIGURE: Von mises stress distribution for each conductor segment

DISPLACEMENT FIELD, $\sigma(T), k(T)$

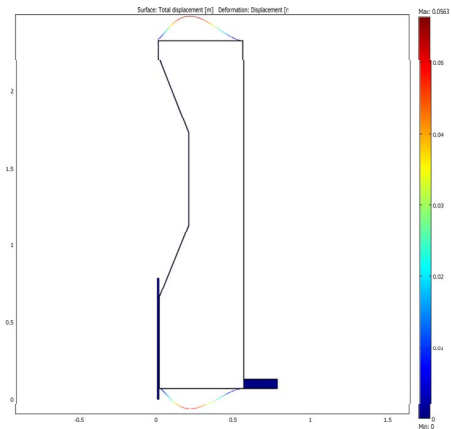


FIGURE: Total displacement due to magnetic and thermal stress, $U_{max} = 5$ cm

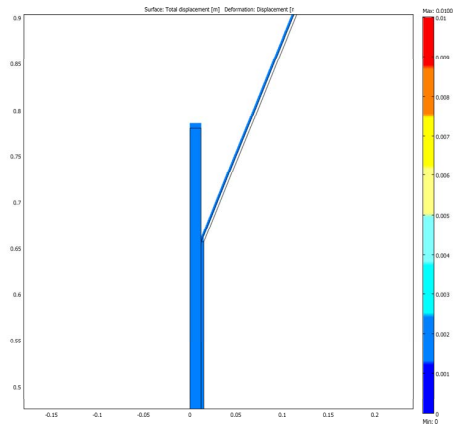
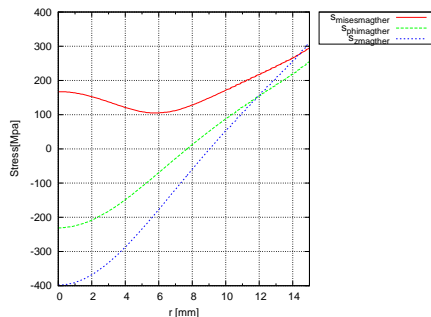


FIGURE: Total displacement in the target/conductor region, $U_{max} \sim 2, 3$ mm

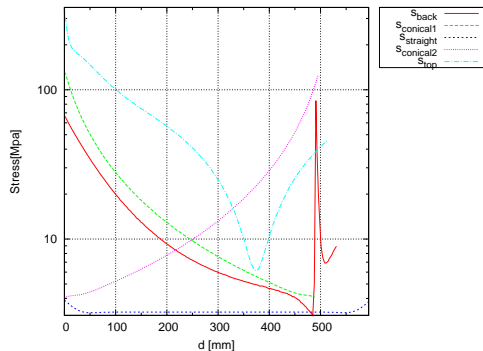
TARGET STRESS, $\sigma(T)$, $k(T)$



- cylinder in traction in z direction for $r \in [0.8, 1.5]$ cm
- stress level higher than fatigue strength (2 times higher than fatigue strength of Be)
- would be (maybe ?) ok if the target was not a structural element of the horn
- for an integrated target: level of stress too high: \Rightarrow increased cooling to decreased thermal stress.

FIGURE: Stress in the radial direction at $z = 2$ cm

HORN STRESS, $\sigma(T)$, $k(T)$



- Stress increase with lower radius
- too high stress level in conical section
- stress level higher than fatigue strength
- difference only in the beryllium part (target) because thermal conductivity of Be changes with temperature, \Rightarrow thermal stress

- $N = 8E8$: total number of pulses
- 4 horns: $\frac{N}{4}$ pulses per horn at frequency 12.5 Hz.
- $\tau = \frac{N}{f} = 16 \times 10^6 \text{s}$, ~ 6 months continuously
- Al: no fatigue limit, properties degrading as N increased
- max stress for AL as low as possible, maybe below 50 Mpa
- fatigue limit of Be: ~ 100 Mpa.
- different story for weld junctions
- Need study on irradiation effect on materials and lifetime.
- Effect of water on lifetime ?

CONCLUSION/SOLUTIONS

- increased thickness and/or increased cooling to decreased thermal stress.
- model with increased thickness
- need to have low stress to have acceptable lifetime.
- others: fatigue joints, welding.
- effect of irradiation; structural damage
- effect of water