



Laboratoire d'Annecy-le-Vieux
de Physique des Particules

First studies of the anode deck structure

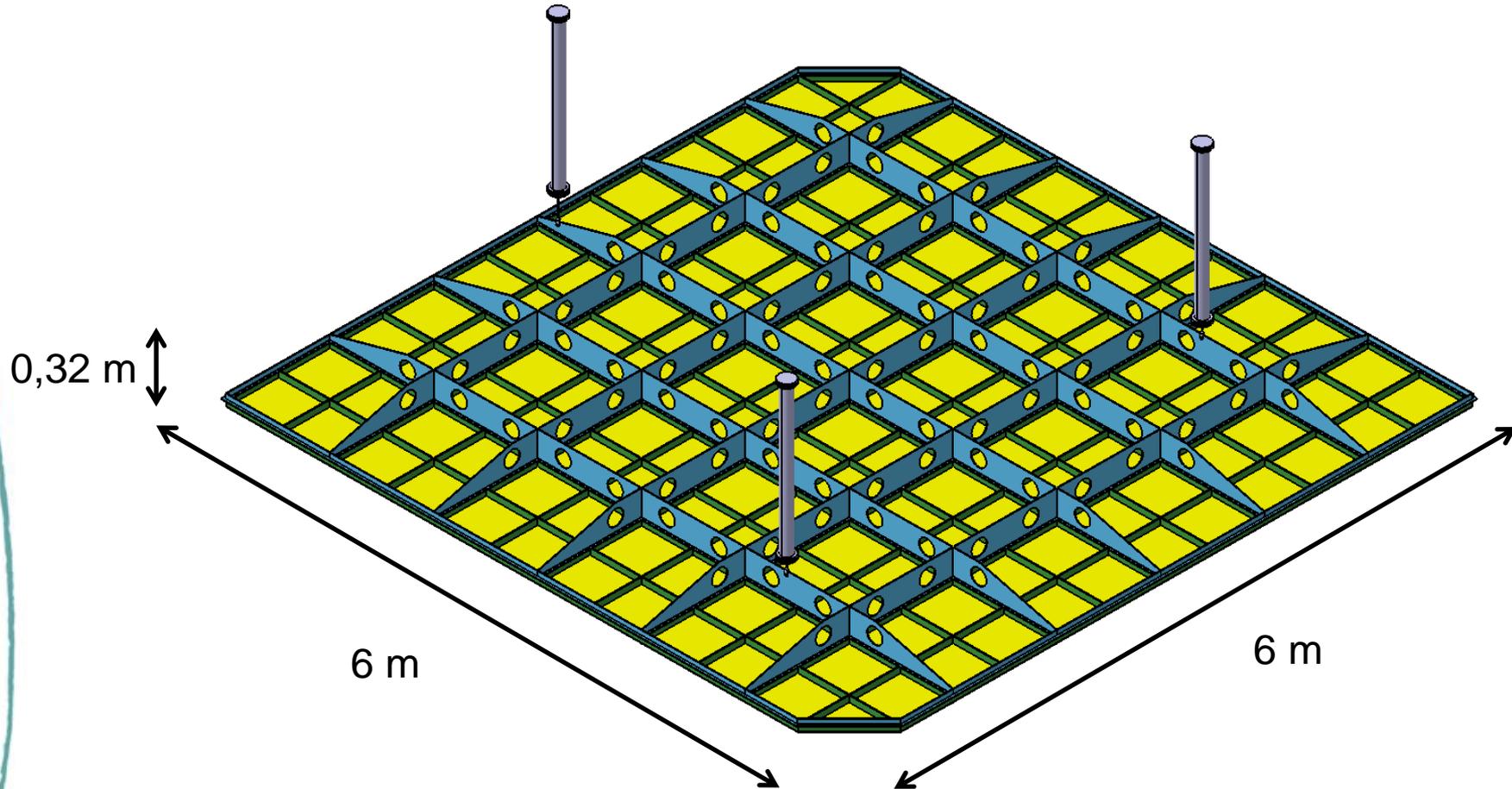
(Thermal and mechanical simulations)

Visio-conference - 27th of February, 2014

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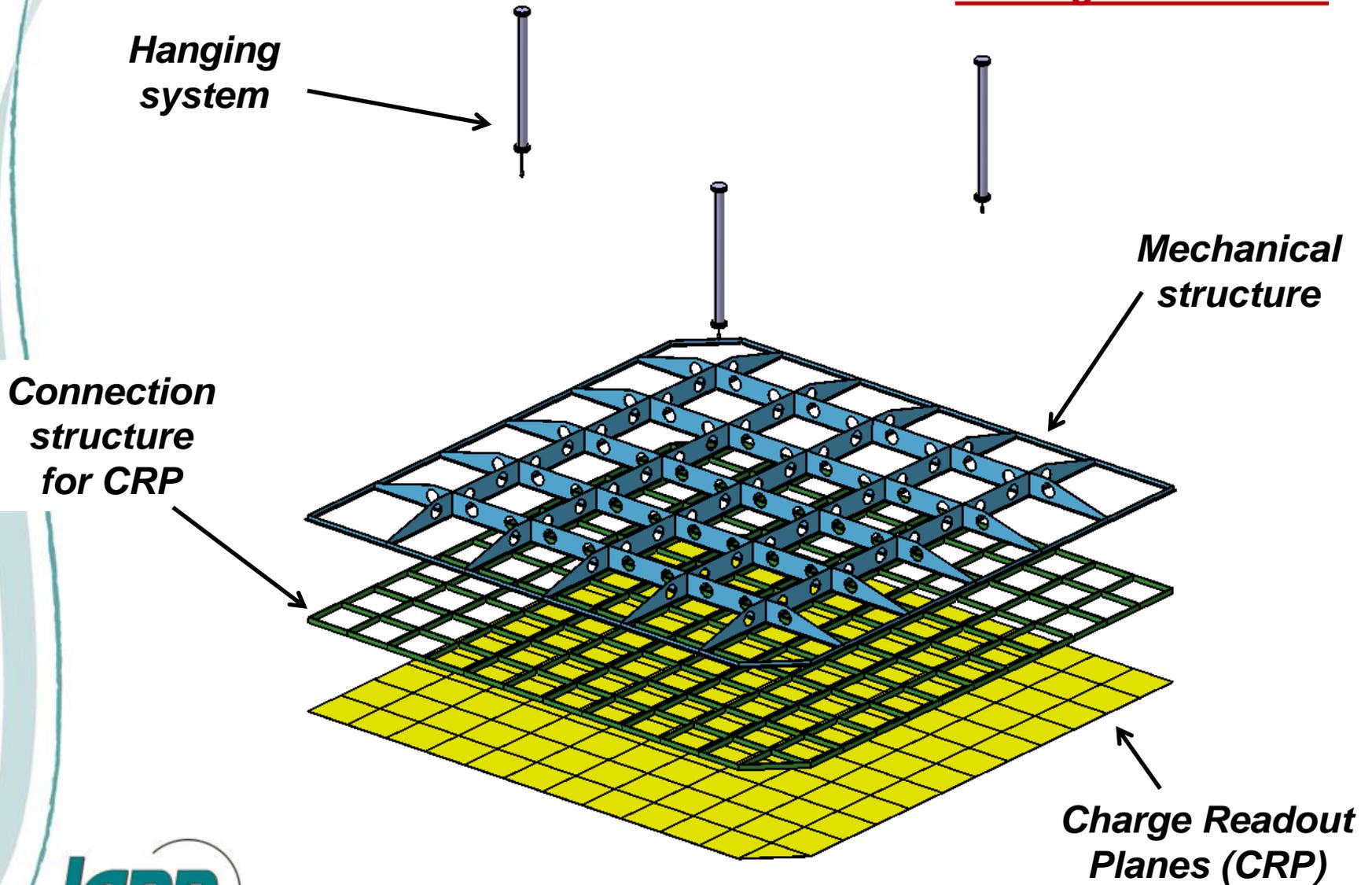
Baseline design

CAD model from Adamo Gendotti



Baseline design

Naming conventions



FEA assumptions 1/2

➤ G10 properties :

- $E = 24.000 \text{ MPa}$
- $\nu = 0,3$
- $\rho = 1850 \text{ kg/m}^3$
- $\alpha = 1,5 \cdot 10^{-5} \text{ K}^{-1}$

NB:

- **E** : Elastic Modulus
- **ν** : Poisson coefficient
- **ρ** : Density
- **α** : Thermal expansion coefficient

➤ Stainless steel properties :

- $E = 210.000 \text{ MPa}$
- $\nu = 0,3$
- $\rho = 7850 \text{ kg/m}^3$
- $\alpha_{304L} = 1,7 \cdot 10^{-5} \text{ K}^{-1}$
- $\alpha_{316L} = 1,6 \cdot 10^{-5} \text{ K}^{-1}$

➤ Added mass :

- 10 kg/m^2 over 36 m^2 for electronics, mesh and tensioning system
- Distributed on CRP

FEA assumptions 2/2

➤ Boundary conditions:

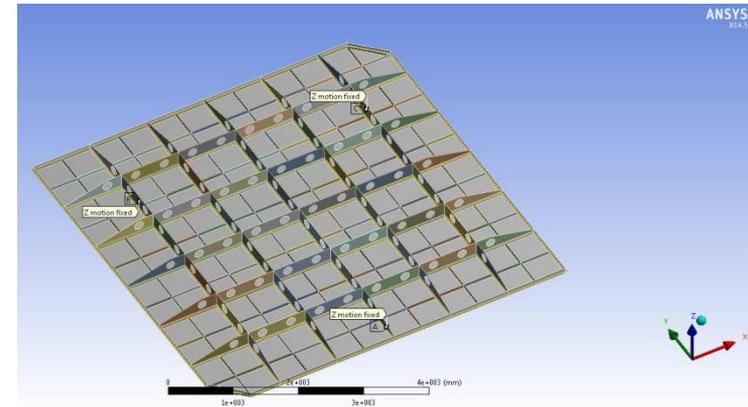
- Z motion fixed for cables anchoring locations
- X and Y motions free for all the structure

➤ Static loading:

- Gravity acting along $-Z$ direction

➤ Thermal loading:

- Thermalization of the structure (same temperature on all nodes)
- From 22°C to -186°C , imposed on the whole model



Comments on assumptions

➤ G10 properties :

- $E = 24.000 \text{ MPa}$
- $\nu = 0,3$
- $\rho = 1850 \text{ kg/m}^3$
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➤ Added mass :

- 10 kg/m^2 over 36 m^2 for electronics, mesh and tensioning system
- Distributed on CRP

➤ G10 considered isotropic

➤ All values (E, ν, α) at 22°C

➤ Large range of α values for stainless steel

➤ Added mass roughly estimated (as suggested)

➤ Mesh tensioning system has no influence on the structure

Goal to reach

The objective is to find a structure whose deformations, occurring during data taking phases, respect the criterion:

“The CRP displacements of any node cannot exceed +/- 0,5 mm along the vertical direction”

(ie perpendicularly to the LAr free surface)

Which is similar to:

The peak to peak value ||Dz|| of the CRP displacements must fulfil :

$$\underline{\underline{||Dz|| < 1mm}}$$

Simulation 1

Short description:

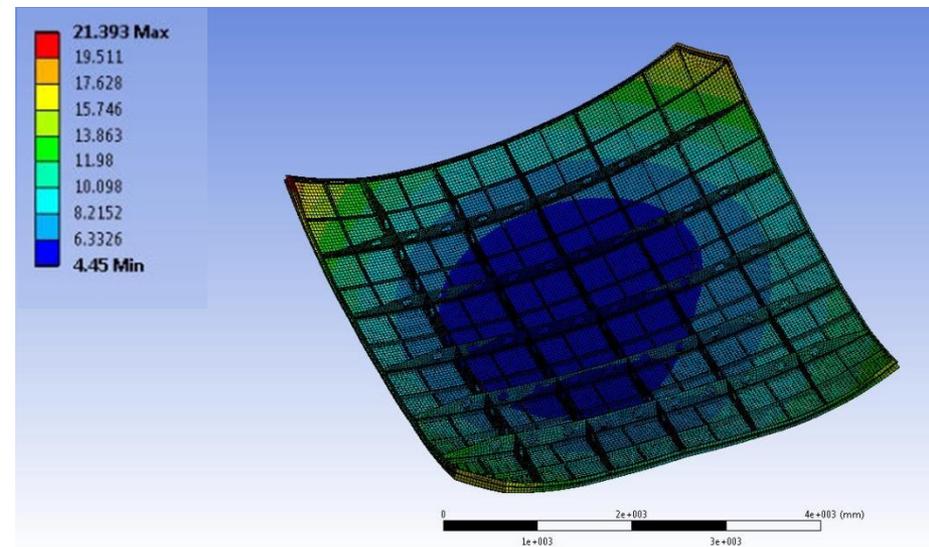
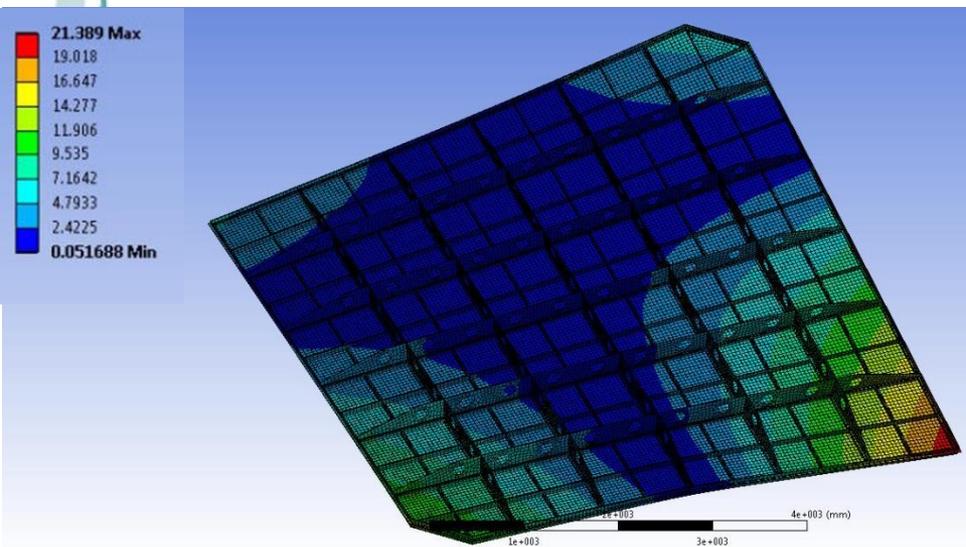
- Baseline design
- CRP and Connection structure made of G10
- Mechanical structure made of 316L stainless steel
(closest thermal expansion coefficient wrt G10 one)
- Hanging system = 3 cables

Simulation 1

Amplitude of 3 displacements (dX, dY, dZ)

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



Huge change of shape !

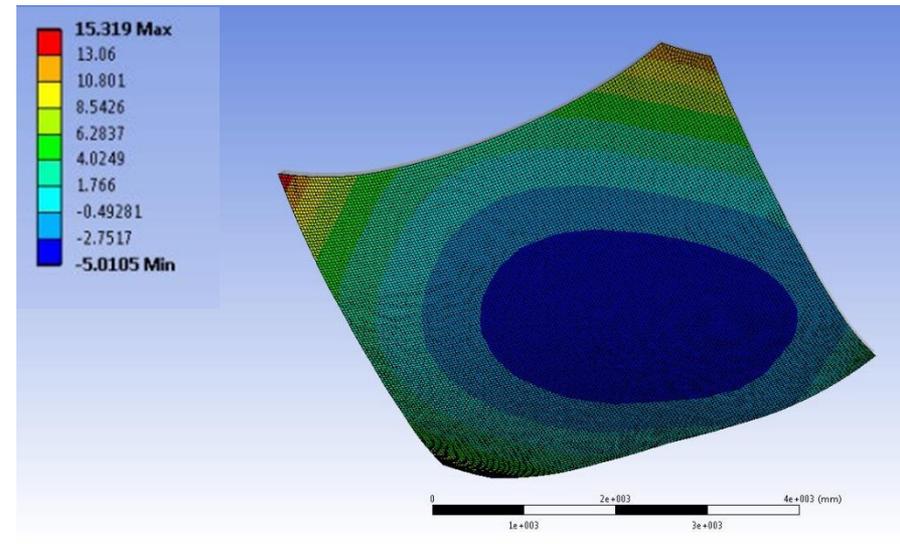
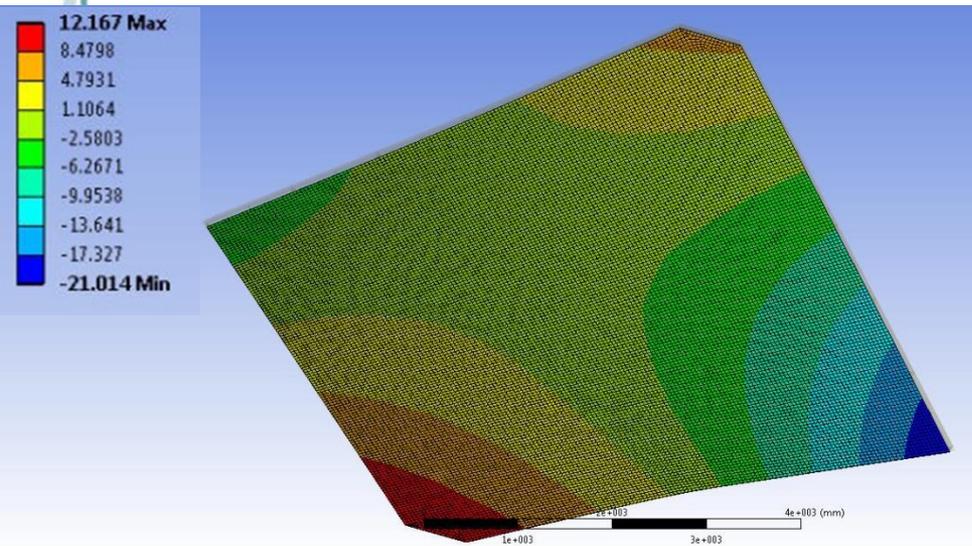
Non negligible influence of *thermal bi-material effect*

Simulation 1

Displacements of CRP along Z direction

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



	Step 1		Step 2		
	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	Model mass
<i>Simulation 1</i> G10 + SSteel / 3 cables	[-21 ; 12,2]	33,2 mm	[-5 ; 15,3]	20,3 mm (>1mm)	1868 kg

Simulation 2

Short description:

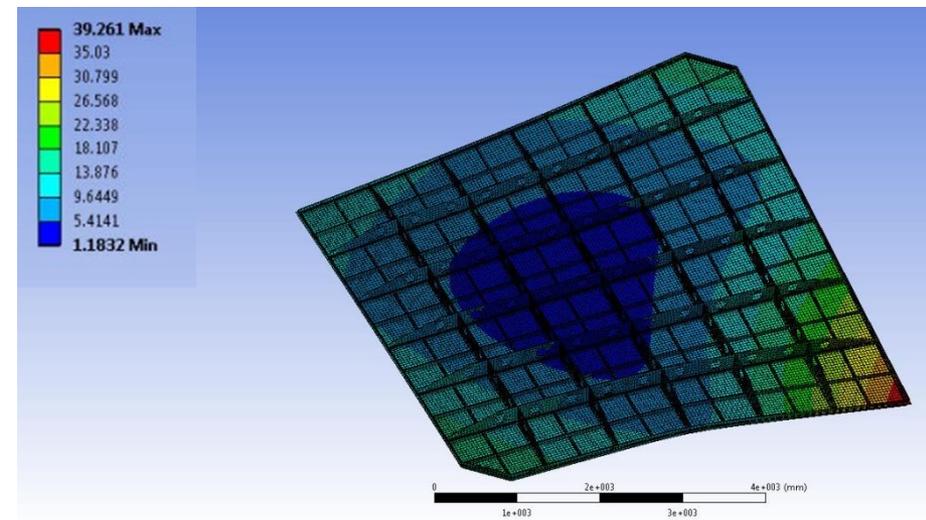
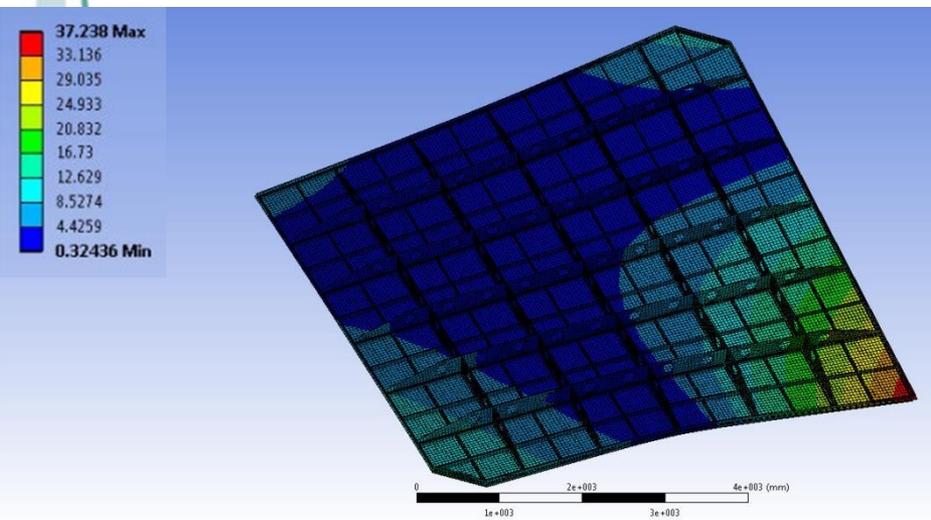
- Baseline design
- All the structure made of G10
- Hanging system = 3 cables

Simulation 2

Amplitude of 3 displacements (dX, dY, dZ)

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



No change of shape

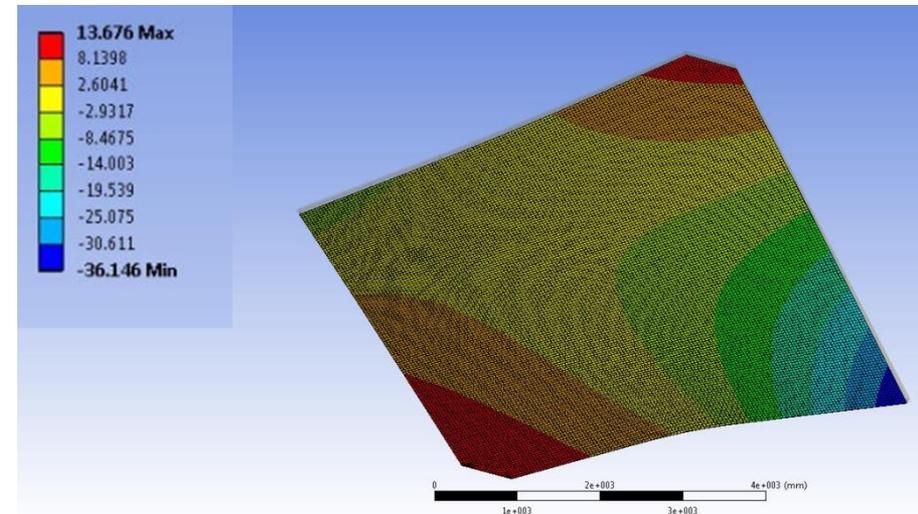
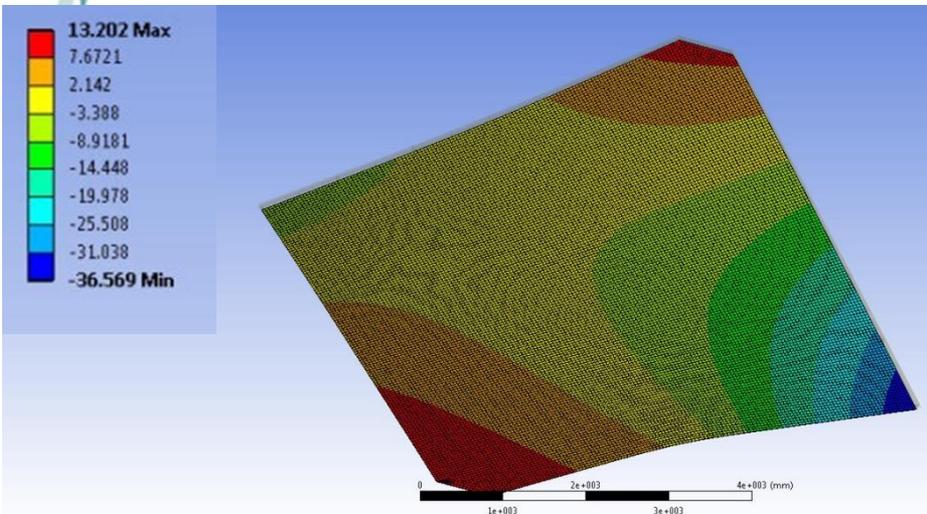
Thermal effect negligible on this full G10 structure

Simulation 2

Displacements of CRP along Z direction

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



	Step 1		Step 2		
	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	Model mass
<i>Simulation 2</i> G10 / 3 cables	[-36,6 ; 13,2]	49,8 mm	[-36,2 ; 13,7]	<u>49,9 mm</u> (> 1 mm)	1167 kg

First comparison

Step 1:
gravity ONLY

Step 2:
gravity + ΔT (+22° to -186°C)

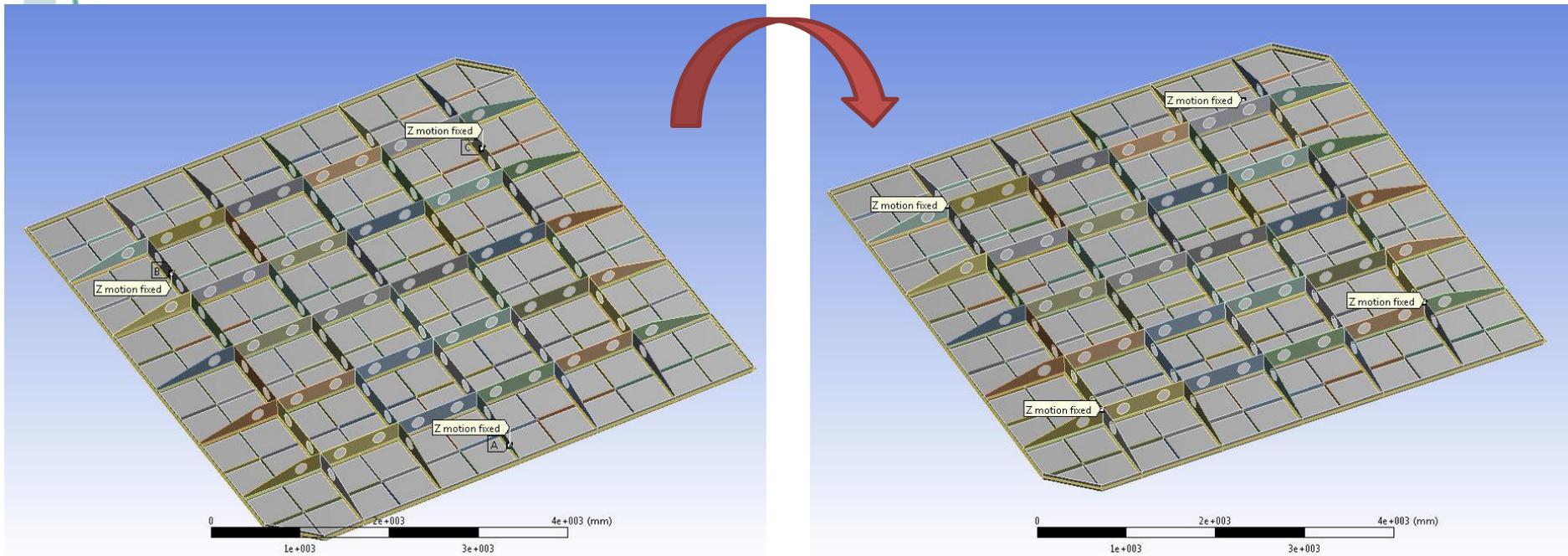
	Step 1		Step 2		
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- Full G10 structure is obviously less stiff than G10 + Stainless steel
- Full G10 structure has a “neutral” behavior on thermal loading
(CRP deformations = in plane deformations !)

Change of boundary conditions

Let's change the boundary conditions:

- by using 4 cables instead of 3
- by fixing these cables at the tips of a square



Simulation 3

Short description:

- Baseline design
- CRP and Connection structure made of G10
- Mechanical structure made of 316L stainless steel (closest thermal expansion coefficient wrt G10 one)
- **Hanging system = 4 cables**

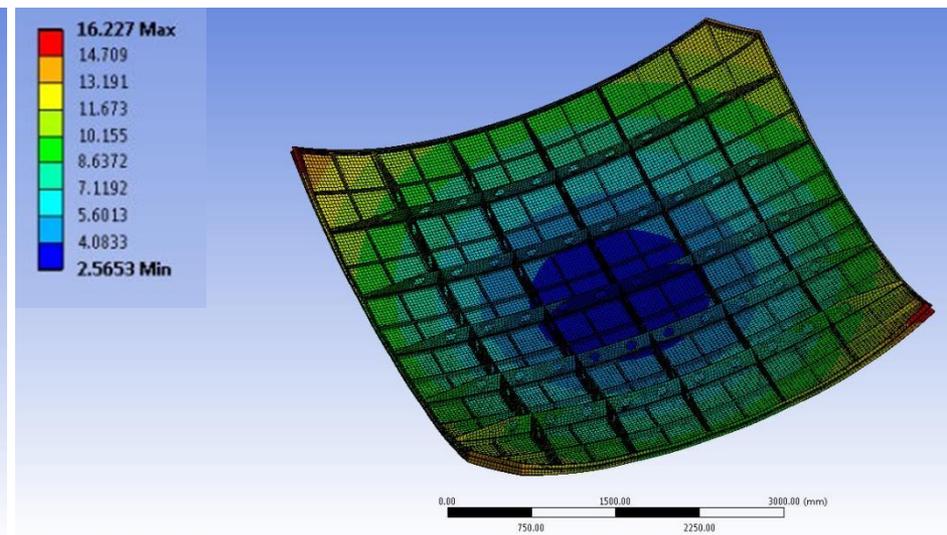
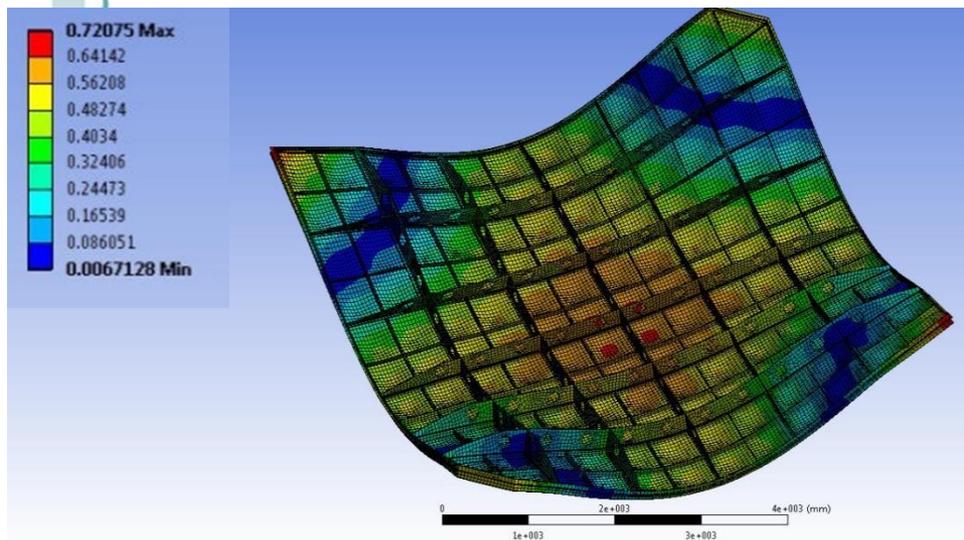
NB : same than simulation 1 with 1 more cable !

Simulation 3

Amplitude of 3 displacements (dX, dY, dZ)

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



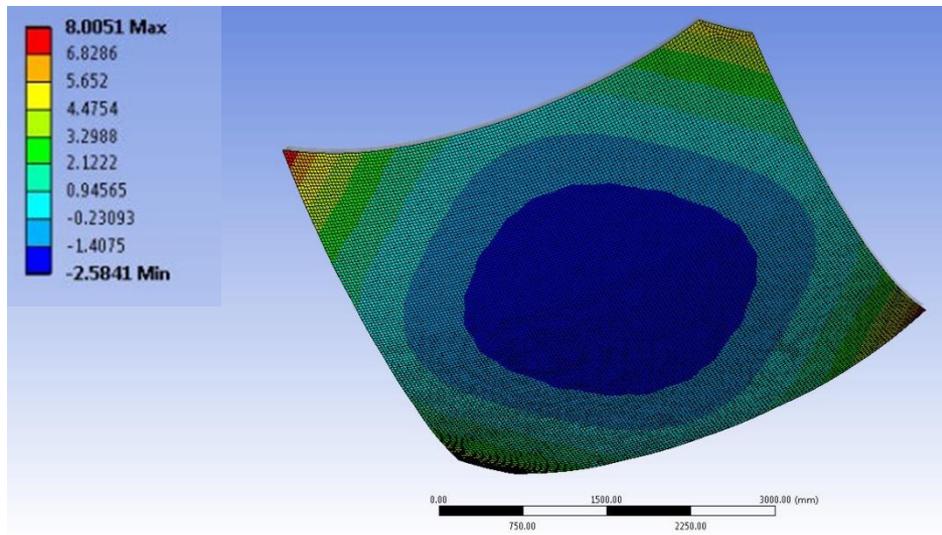
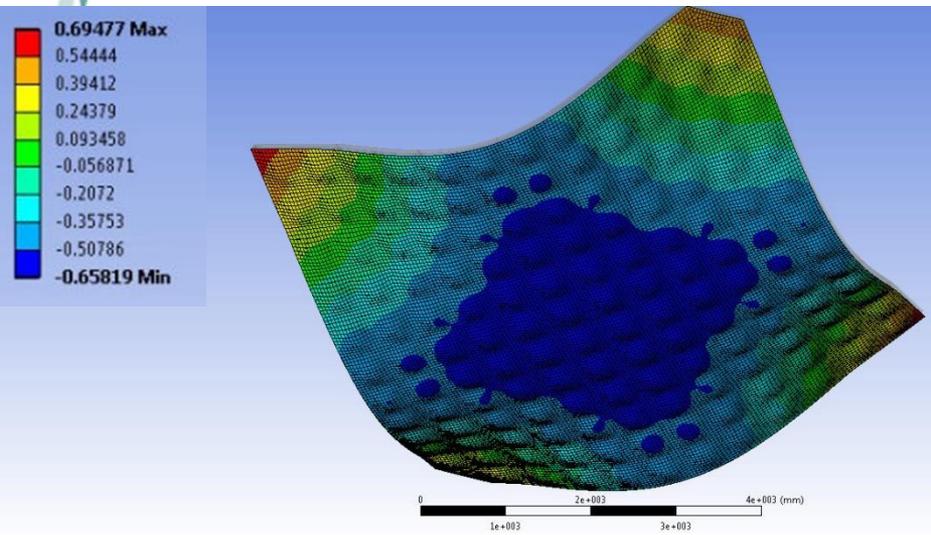
Big reduction of displacements
(compared to the same solution with 3 cables)

Simulation 3

Displacements of CRP along Z direction

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



	Step 1		Step 2		
	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	Model mass
<i>Simulation 3</i> G10 + SSteel / 4 cables	$[-0,7 ; 0,7]$	1,4mm	$[-2,6 ; 8]$	10,6 (> 1 mm)	1868 kg

Still thermal bi-material effect non negligible

Simulation 4

Short description:

- Baseline design
- All the structure made of G10
- Hanging system = 4 cables

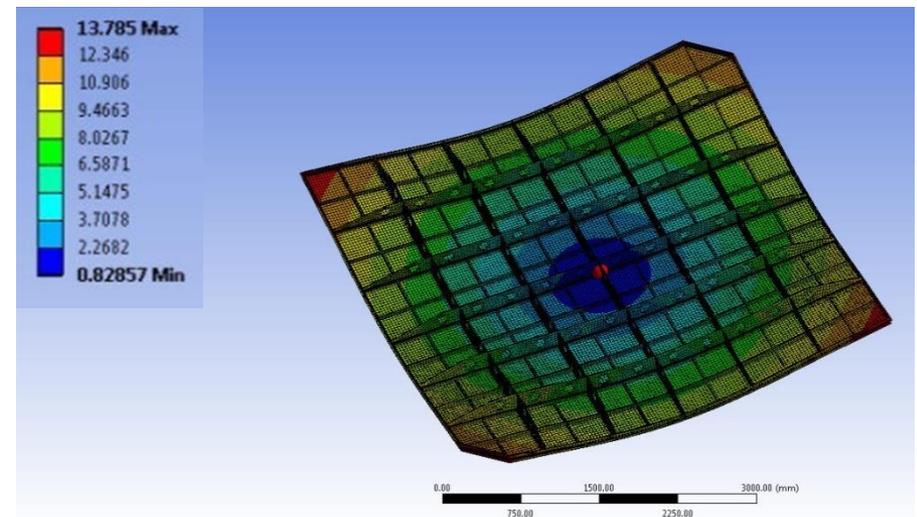
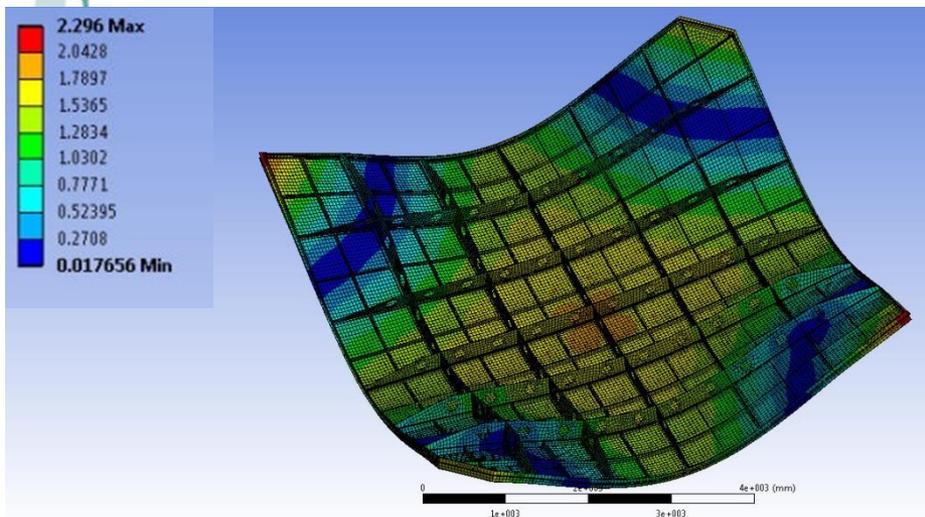
NB : same than simulation 2 with 1 more cable !

Simulation 4

Amplitude of 3 displacements (dX, dY, dZ)

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



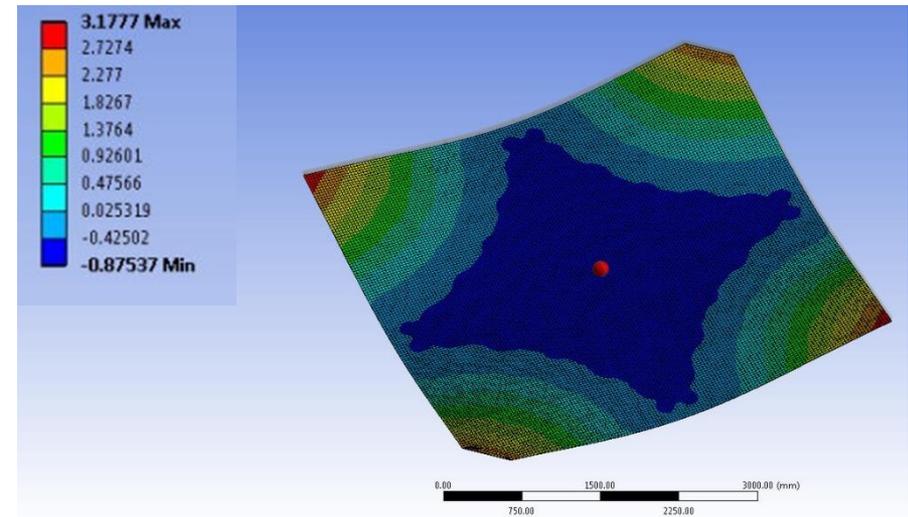
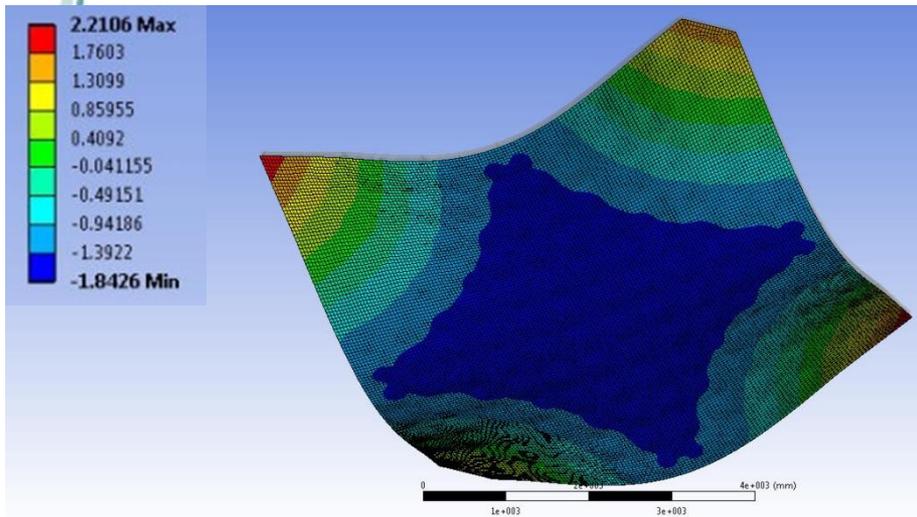
Big reduction of displacements
(compared to the same solution with 3 cables)

Simulation 4

Displacements of CRP along Z direction

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



	Step 1		Step 2		
	$[Dz_{min}; Dz_{max}]$	$\ Dz\ $	$[Dz_{min}; Dz_{max}]$	$\ Dz\ $	Model mass
<i>Simulation 4</i> G10 / 4 cables	[-1,8 ; 2,2]	4mm	[-0,9 ; 3,2]	4,1 (> 1 mm)	1167 kg



“neutral” behavior on thermal loading

Second comparison

Step 1:
gravity ONLY

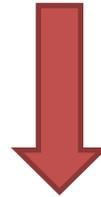
Step 2:
gravity + ΔT (+22° to -186°C)

	Step 1		Step 2		
	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	Model mass
<u>Simulation 1</u> G10 + SSteel / 3 cables	[-21 ; 12,2]	33,2mm	[-5 ; 15,3]	<u>20,3</u>	1868 kg
<u>Simulation 2</u> G10 / 3 cables	[-36,6 ; 13,2]	49,8mm	[-36,2 ; 13,7]	<u>49,9 mm</u>	1167 kg
<u>Simulation 3</u> G10 + SSteel / 4 cables	[-0,7 ; 0,7]	1,4mm	[-2,6 ; 8]	<u>10,6</u>	1868 kg
<u>Simulation 4</u> G10 / 4 cables	[-1,8 ; 2,2]	4mm	[-0,9 ; 3,2]	<u>4,1</u>	1167 kg

- 4 cables help a lot to drastically decrease the CRP displacements
- Full G10 structure seems to be a promising option

Comments

- Full G10 structure has a very interesting thermal behavior
- It is nevertheless less stiff than associated with stainless steel



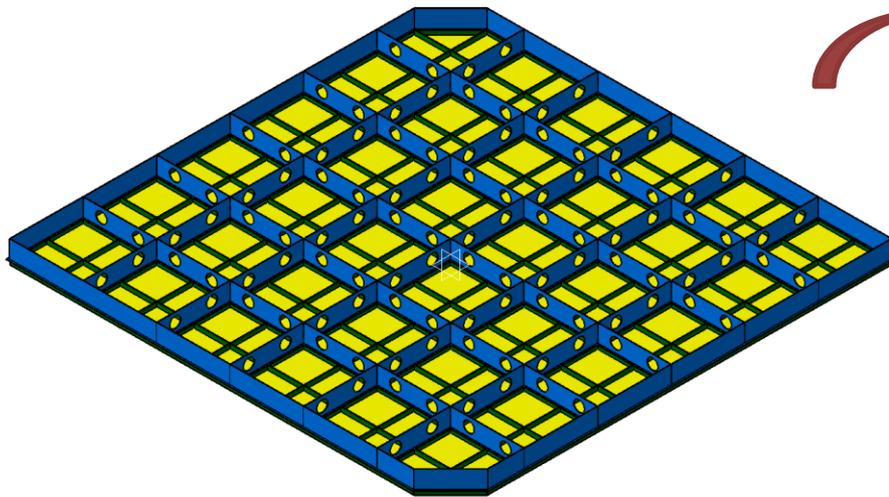
Why not keep the full G10 option
&
try to find a stiffer design (for static loading) ?

New design proposal

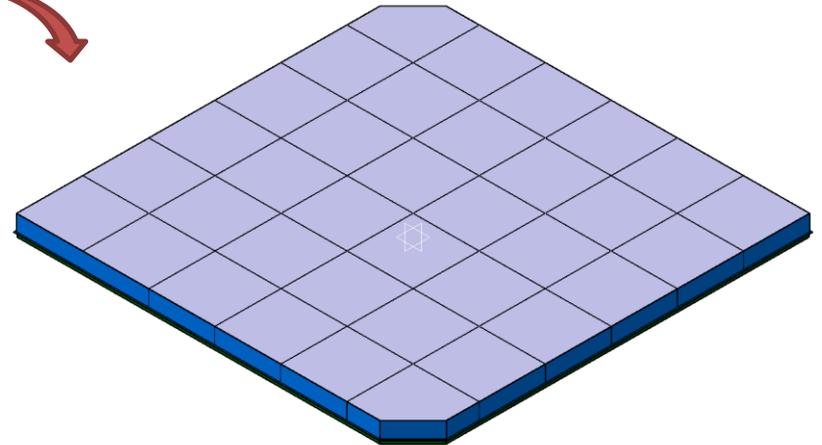
Based on “Adamo design” (same structure thickness), the use of a “closing plate” could improve the static behavior of the structure

Slight modifications

(Belt surrounding the structure)



Closing plate added



Simulation 5

Short description:

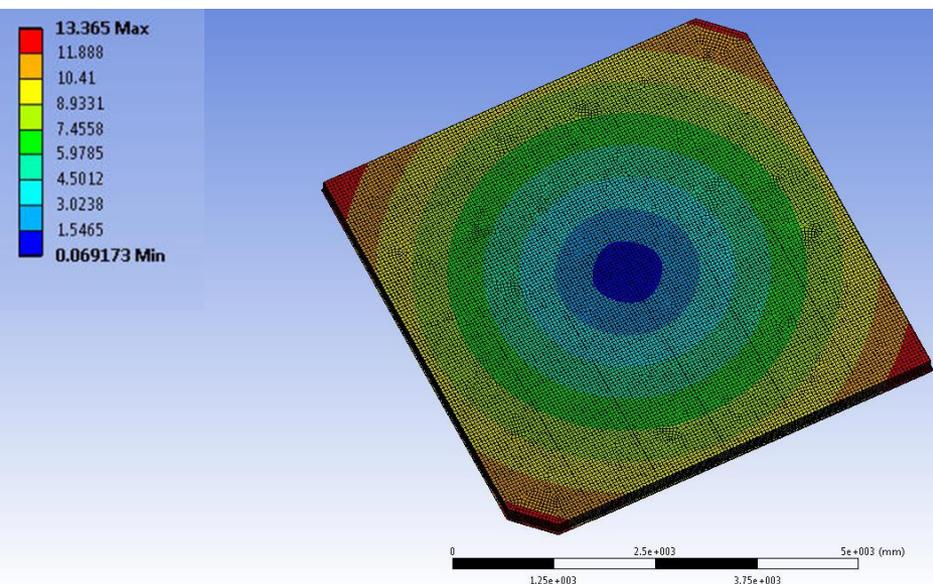
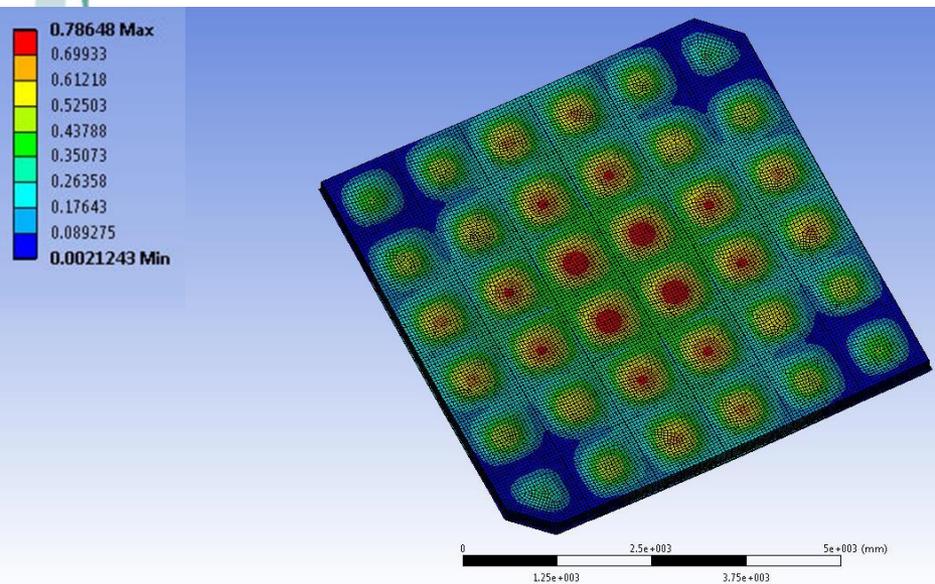
- Baseline design + belt + closing plate
- All the structure made of G10
- Hanging system = 4 cables

Simulation 5

Amplitude of 3 displacements (dX, dY, dZ)

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



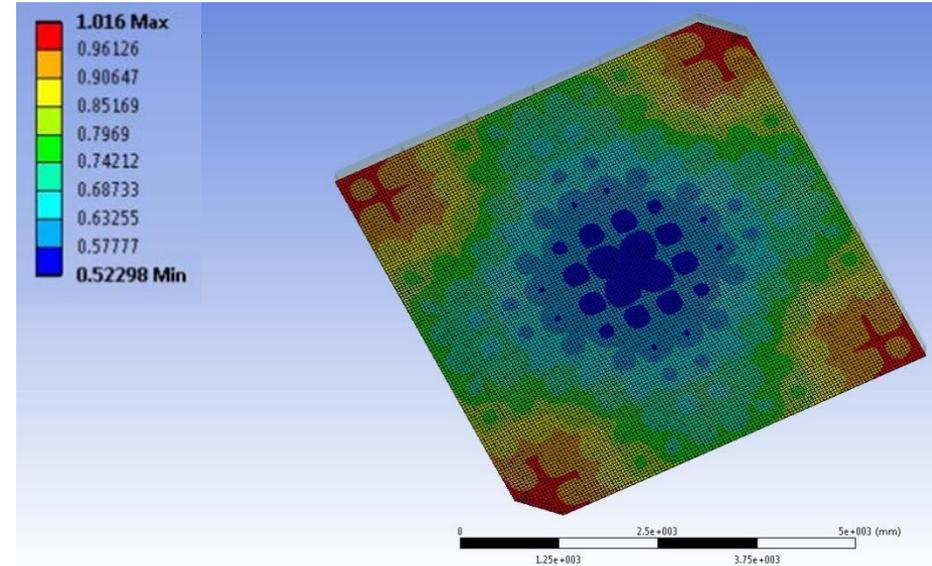
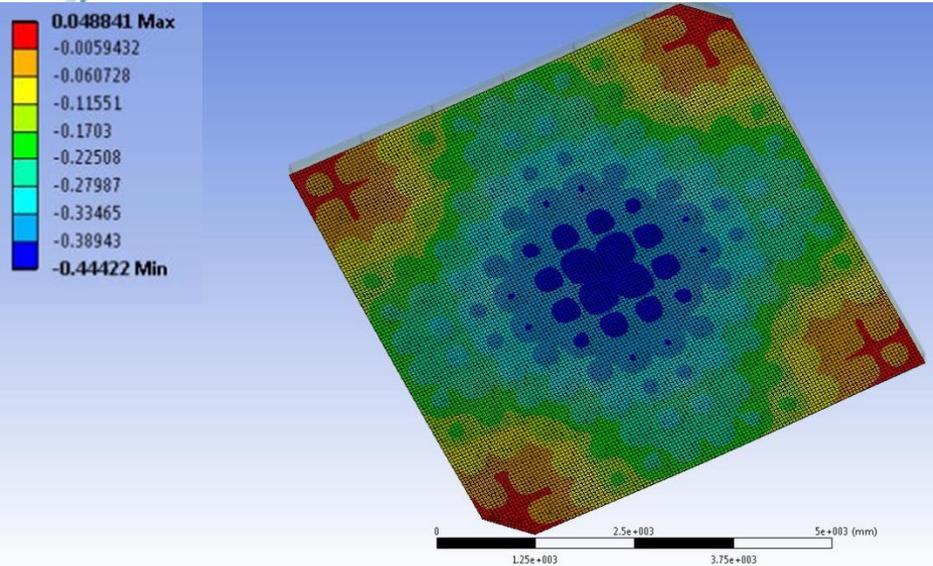
Small static displacements and
homogeneous structure expansion

Simulation 5

Displacements of CRP along Z direction

Step 1: gravity ONLY

Step 2: gravity + ΔT (+22° to -186°C)



	Step 1		Step 2		
	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	$[Dz_{min} ; Dz_{max}]$	$\ Dz\ $	Model mass
<i>Simulation 5</i> G10 / 4 cables / closing plate	$[-0,44 ; 0,05]$	0,5mm	$[0,5 ; 1]$	<u>0,5</u>	1600 kg



This model fulfills the “1mm max” requirement !

Advanced thermal simulations

In the previous simulations, the thermal loading is extremely simple:
+22°C to -186 °C is applied to all the structure.

In reality, a thermal gradient can happen due to:

- Temperature variation as a function of altitude / LAr free surface: $\Delta T = f(z)$?
- Thermal flux (heat source) coming from electronics & power/data cables ?
- Thermal flux (heat source) coming from chimneys / supporting cables ?
- Thermal radiation from insulating walls ?

A more realistic simulation would take into account these loadings

Advanced thermal simulations

As a result a thermal gradient would appear in the structure

In order to illustrate the behavior of the structure, a temperature gradient has been applied thanks to a surface flux.

This thermal loading is arbitrary.
It will nevertheless help us to understand how behaves the anode deck structure.

Note that:

- The arbitrary surface flux is applied to the closing plane (to simplify).
- The flux is a parameter which varies between $[0 ; 1.10^{-7} \text{ W/mm}^2]$
- For zero flux, the results are the same than for simulation #5.

Advanced thermal simulations

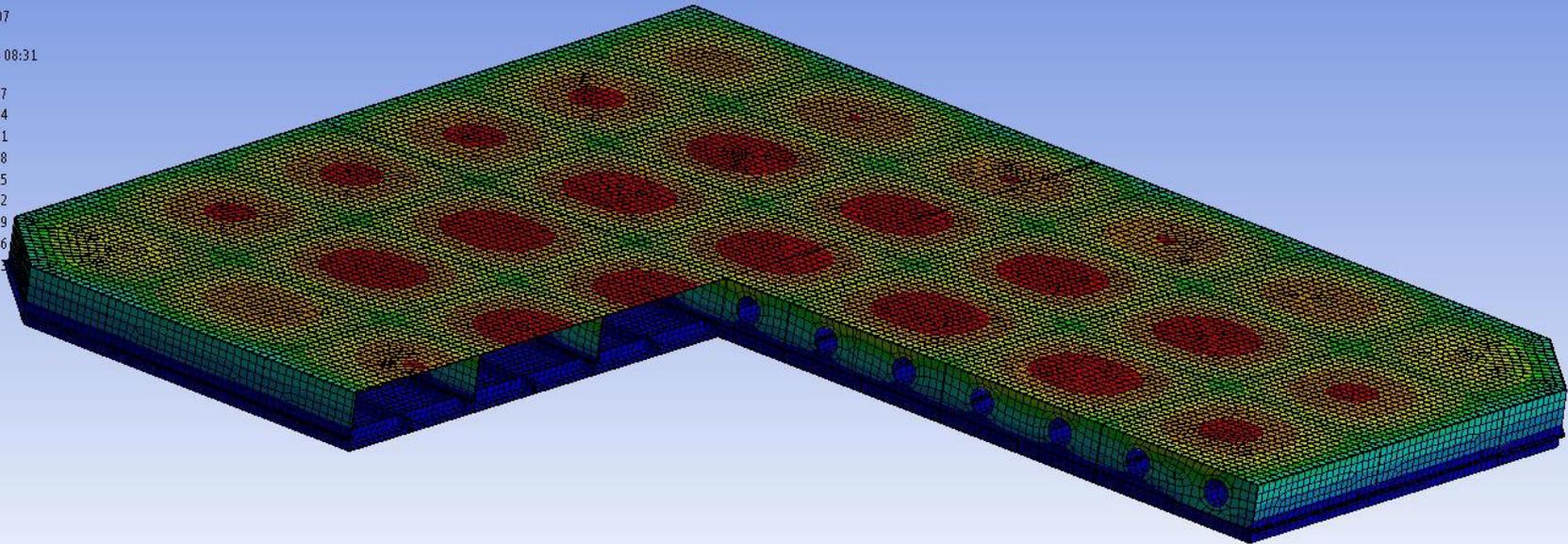
For the max. heat flux, we observe a 16°C gradient

It is maybe overestimated but it is useful to understand this parametric study

ANSYS
R14.5

B: Steady-State Thermal
Temperature
Type: Temperature
Unit: °C
Time: 1
Custom
Max: -170.07
Min: -186
26/02/2014 08:31

-170.07
-171.84
-173.61
-175.38
-177.15
-178.92
-180.69
-182.46
-184.23
-186



0.00 500.00 1000.00 1500.00 2000.00 (mm)

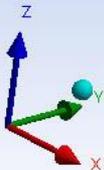
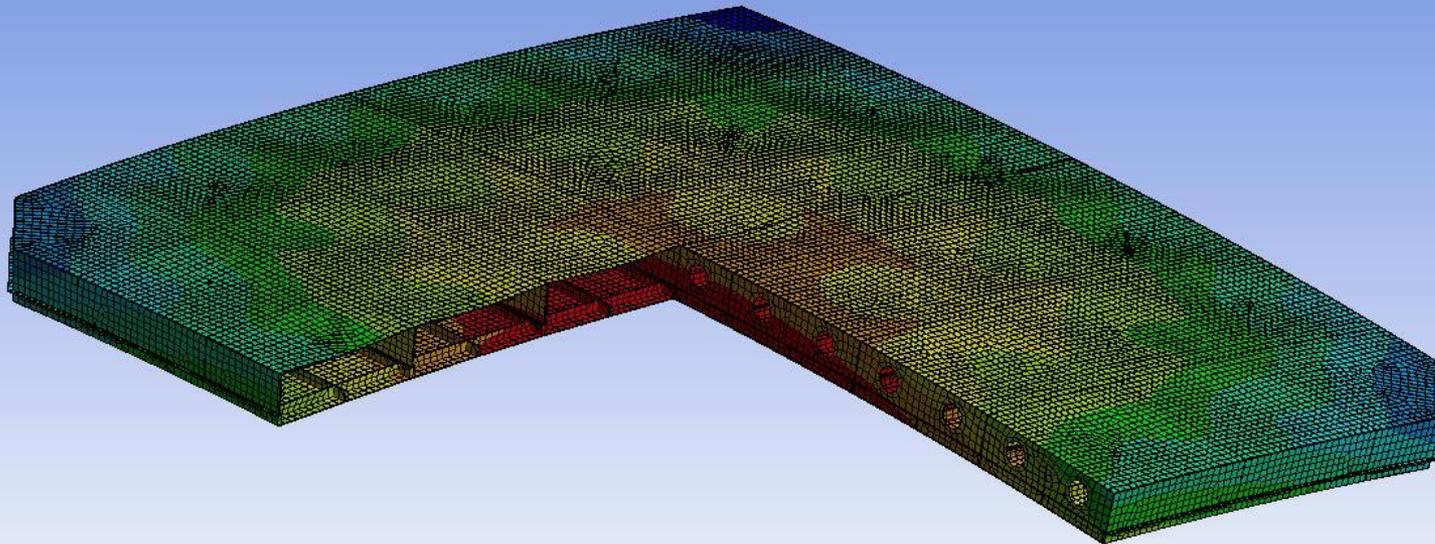
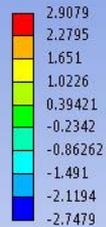
NB: Cutting planes are used to illustrate the temperature gradient.

Advanced thermal simulations

Structure displacements resulting from the 16°C gradient

ANSYS
R14.5

C: Static Structural
Directional Deformation
Type: Directional Deformation(Z Axis)
Unit: mm
Global Coordinate System
Time: 1
Max: 2.9079
Min: -2.7479
26/02/2014 08:32



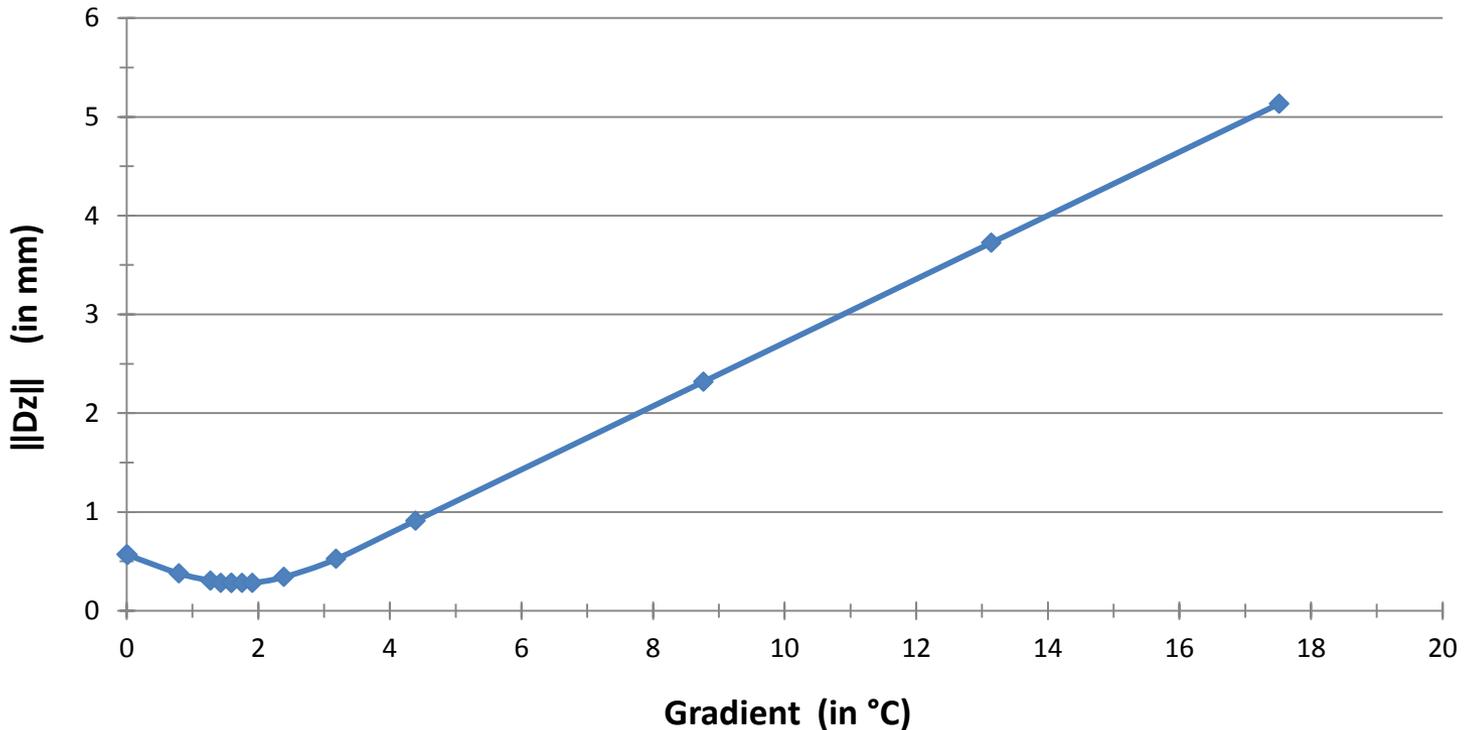
5mm peak to peak displacement for the CRP



Advanced thermal simulations

By varying the heat flux ($[0 ; 1.10^{-7} \text{ W/mm}^2]$) to get a given gradient, we calculate the peak to peak displacements of CRP

$\|Dz\|$ as a function of the thermal gradient

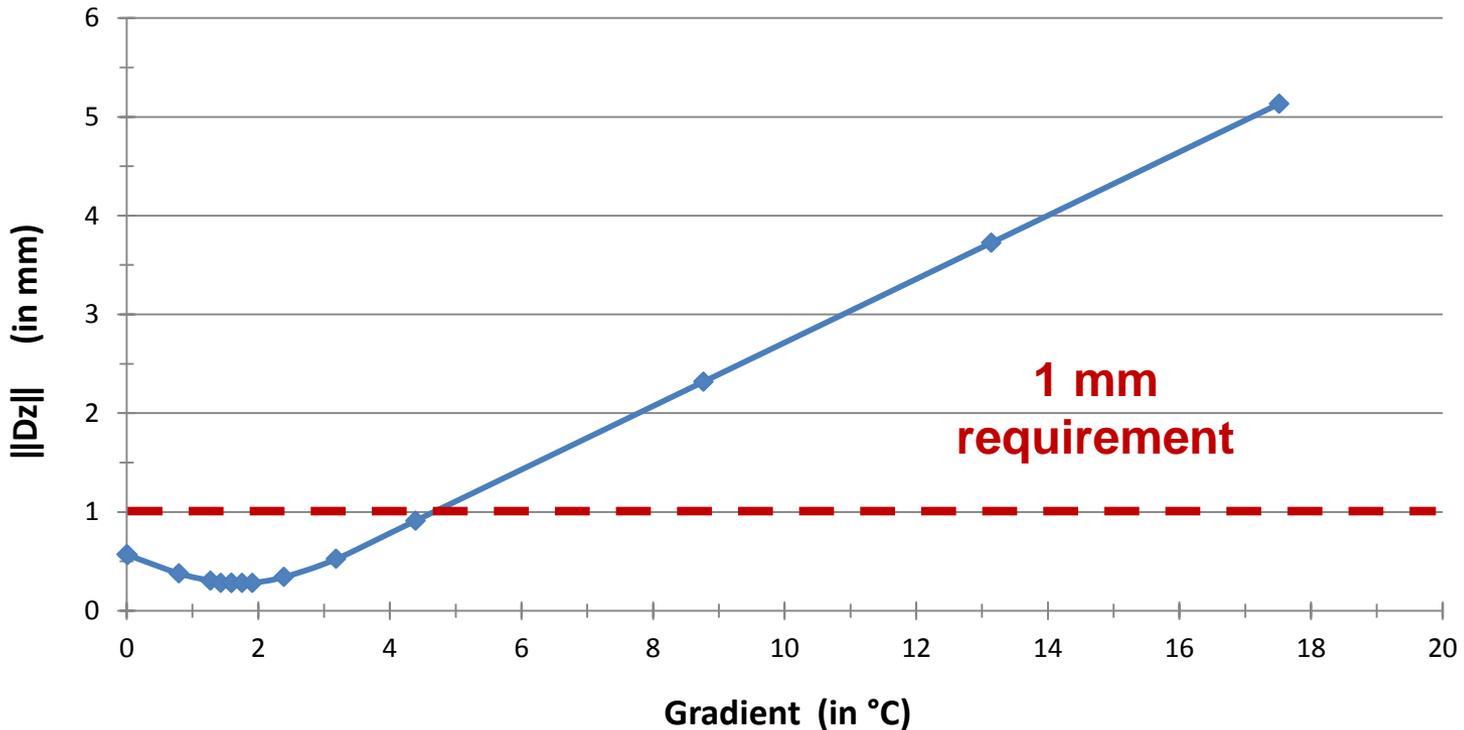


NB: the max heat flux (1.10^{-7} W/mm^2) leads to **3,6 W**

Advanced thermal simulations

By varying the heat flux ($[0 ; 1.10^{-7} \text{ W/mm}^2]$) to get a given gradient, we calculate the peak to peak displacements of CRP

$\|Dz\|$ as a function of the thermal gradient



4,5°C gradient would lead to the max allowable value of 1 mm !

Conclusions 1 /2

For the TDR:

We would like to detail the model corresponding to simulation #5 which fulfils the 1mm requirement

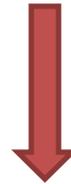
provided that all the structure is thermalized to the same temperature !

Simulation #5 :

- Baseline design + belt + closing plate
- All the structure made of G10
- Hanging system = 4 cables

Conclusions 2 /2

**Structure is sensitive
to thermal gradient**



For the next studies:

We have to discuss about thermal inputs

- to go further into details
- to implement realistic loadings.